

# Control – S-domain



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# Laplace Transform

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## □ Laplace transform

- Laplace transform is named after Pierre-Simon Laplace, who introduced the transform in his work on probability theory.
- Laplace transform simplifies the process of analyzing the behavior of the system. In engineering applications, normally refer to s-domain, which corresponding to a linear time-invariant (LTI) system for system stability and dynamic analysis.
- Laplace transform definition

$$F(s) = \int_0^{\infty} e^{-st} f(t) dt$$

where  $s = \sigma + j\omega$

- Reference:  
[http://en.wikipedia.org/wiki/Laplace\\_transform](http://en.wikipedia.org/wiki/Laplace_transform)

## Mathematical Relationship

### Commonly Used in Electronics

$$f(t) \leftrightarrow F(s)$$

$$\frac{df(t)}{dt} \leftrightarrow sF(s)$$

$$\int f(t) dt \leftrightarrow \frac{1}{s} F(s)$$

### Initial value theorem :

$$f(0^+) = \lim_{s \rightarrow \infty} sF(s)$$

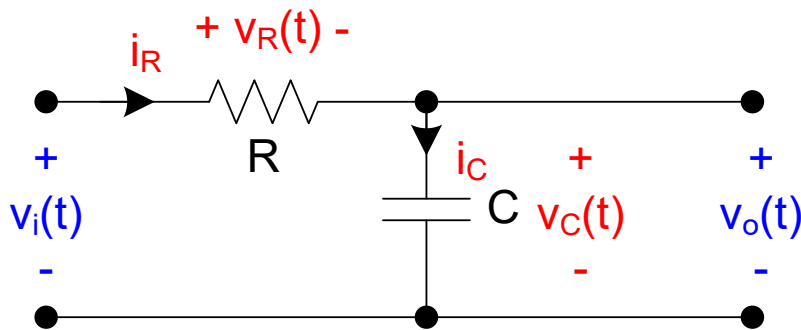
### Final value theorem :

$$f(\infty) = \lim_{s \rightarrow 0} sF(s)$$

# Example of Laplace Transform in electronics circuit

Example of RC Filter

Question: Calculation  $V_o(s)/V_i(s)$



$$(1): i_R = i_C$$

$$(2): i_C = C \frac{dv_C(t)}{dt} = C \frac{dv_o(t)}{dt}$$

For

$$v_i(t) = v_R(t) + v_C(t)$$

$$v_i(t) = i_R R + v_o(t)$$

$$v_i(t) = i_C R + v_o(t)$$

$$v_i(t) = CR \frac{dv_o(t)}{dt} + v_o(t)$$

Laplace Transform

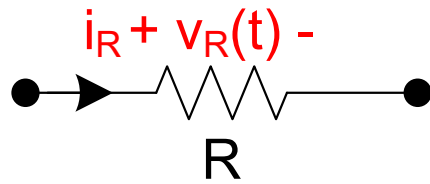
$$V_i(s) = sCRV_o(s) + V_o(s)$$

$$\frac{V_o(s)}{V_i(s)} = \frac{1}{sCR + 1}$$

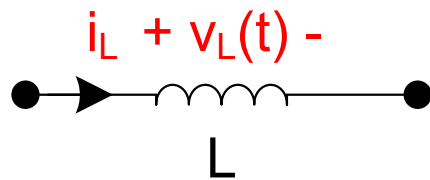
Question: Do we need to setup differential equation first???

# S-domain representation for circuit element

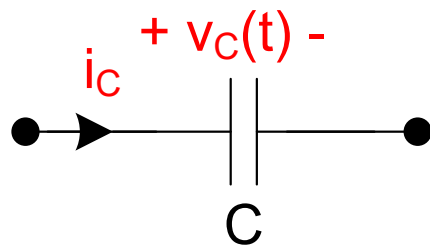
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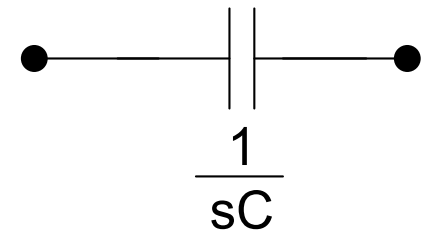
$$v_R(t) = Ri_R(t) \leftrightarrow V_R(s) = RI_R(s)$$
$$\therefore \frac{V_R(s)}{I_R(s)} = R$$



$$v_L(t) = L \frac{di_L(t)}{dt} \leftrightarrow V_L(s) = sLI_L(s)$$
$$\therefore \frac{V_L(s)}{I_L(s)} = sL$$

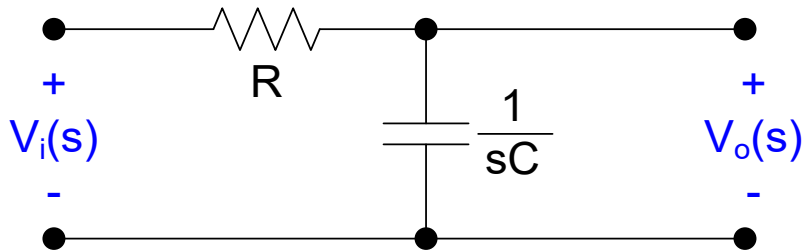


$$v_C(t) = \frac{1}{C} \int i_C(t) dt \leftrightarrow V_C(s) = \frac{1}{sC} I_C(s)$$
$$\therefore \frac{V_C(s)}{I_C(s)} = \frac{1}{sC}$$



# Revisit RC example

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$$V_o(s) = \frac{\frac{1}{sC}}{R + \frac{1}{sC}} V_i(s)$$
$$\frac{V_o(s)}{V_i(s)} = \frac{1}{1 + sCR}$$

Benefit: Simplify circuit analysis without differential equation!

# What can we do with the s-domain transfer function

Example: RC Filter

$$\frac{V_o(s)}{V_i(s)} = \frac{1}{sCR + 1} \rightarrow V_o(s) = \frac{1}{sCR + 1} V_i(s)$$

If input is assumed to be unit step, i.e.  $V_i(t) = \frac{1}{s}$

Apply Initial value theorem :

$$f(0^+) = \lim_{s \rightarrow \infty} sF(s) = \lim_{s \rightarrow \infty} \frac{s}{sCR + 1} V_i(s)$$

$$f(0^+) = \lim_{s \rightarrow \infty} \frac{s}{sCR + 1} \frac{1}{s} = \lim_{s \rightarrow \infty} \frac{1}{sCR + 1} = 0$$

Apply Final value theorem

$$f(\infty) = \lim_{s \rightarrow 0} sF(s) = \lim_{s \rightarrow 0} \frac{s}{sCR + 1} V_i(s)$$

$$f(\infty) = \lim_{s \rightarrow 0} \frac{s}{sCR + 1} \frac{1}{s} = \lim_{s \rightarrow 0} \frac{1}{sCR + 1} = 1$$

Steady state output independent of C and R

Laplace transform can help to calculate the steady state response without solving complicated differential equation.

Input Test Signal and Corresponding Laplace s-domain

Function	Time domain $f(t) = \mathcal{L}^{-1}\{F(s)\}$	Laplace s-domain $F(s) = \mathcal{L}\{f(t)\}$
unit impulse	$\delta(t)$	1
delayed impulse	$\delta(t - \tau)$	$e^{-\tau s}$
unit step	$u(t)$	$\frac{1}{s}$
delayed unit step	$u(t - \tau)$	$\frac{e^{-\tau s}}{s}$
ramp	$t \cdot u(t)$	$\frac{1}{s^2}$

# Benefit of s-domain

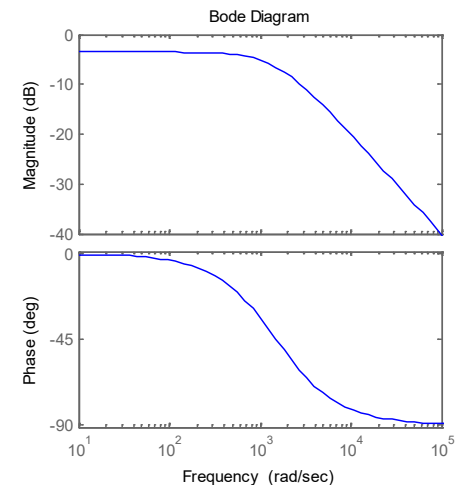
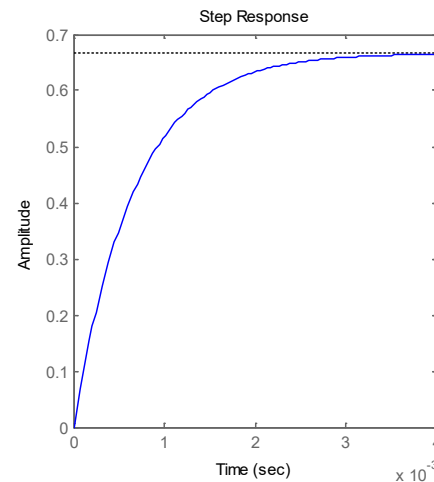
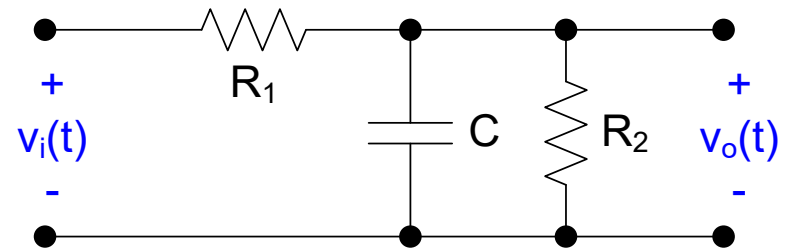
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- In control theory, it well develop the understanding of 1st-order and 2st-order transfer function in s-domain. Therefore, without solving the equation, we can simply conclude the response of a system transfer function without solving the exact equation, and to design proper compensation network. This topic is discussed in another document
  - Control - System Response.doc
  - Control - Matlab and Control.doc

# Homework – Question 1

## Question 1

- Find system transfer function  $G(s) = V_o(s)/V_i(s)$
- Use final value theorem to determine steady state value if unit step response is used.
- Assume  $R_1 = 1k$ ,  $R_2 = 2k$ ,  $C = 1\mu F$ , use matlab to plot the step response and bode plot.
  - You will use matlab function
    - TF
    - STEP
    - BODE
  - Verify the step response with LTspice

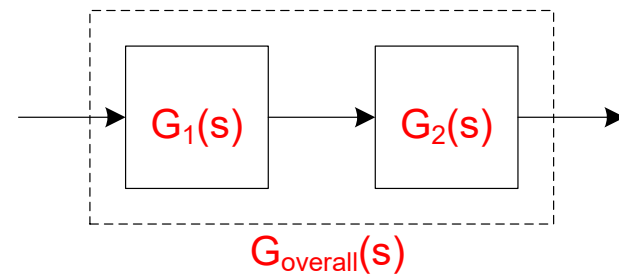
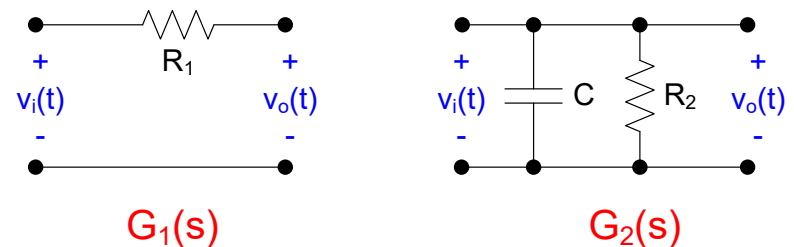




# Homework – Question 2

## □ Question 2

- Calculate system transfer function  $G_1(s)$  and  $G_2(s)$ .
- If  $G_1(s)$  and  $G_2(s)$  are connected in series (cascade), what is the new transfer function  $G_{\text{overall}}(s)$ ?
- Explain why  $G_{\text{overall}}(s)$  is not same as answer of question 1.



# Control – Block Diagram



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# Block Diagram

Close Loop Transfer Function  $\frac{V_{out}}{V_{in}}$

$$\text{Eqn 1: } V_{out} = G(s) \cdot V_{error}$$

$$\text{Eqn 2: } V_f = H(s) \cdot V_{out}$$

$$\text{By } V_{error} = V_{in} - V_f$$

$$\frac{V_{out}}{G(s)} = V_{in} - H(s) \cdot V_{out}$$

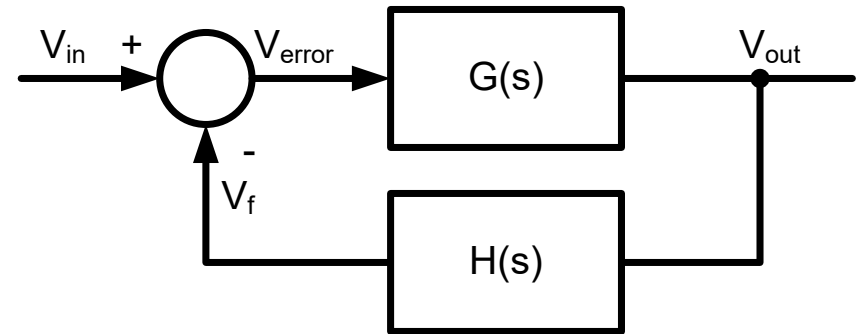
$$V_{in} = \left( \frac{1}{G(s)} + H(s) \right) V_{out} = \frac{1 + G(s)H(s)}{G(s)} \cdot V_{out}$$

$$\frac{V_{out}}{V_{in}} = \frac{G(s)}{1 + G(s)H(s)}$$

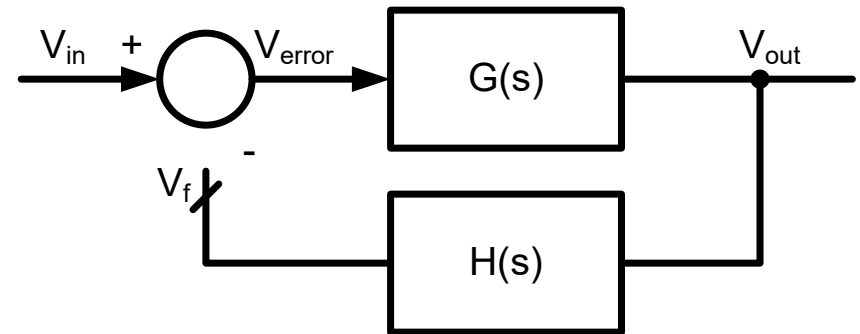
Open Loop Transfer Function  $\frac{V_f}{V_{in}}$

$$\frac{V_f}{V_{in}} = G(s)H(s)$$

Close-Loop Transfer Function



Open-Loop Transfer Function



# Numerical Example

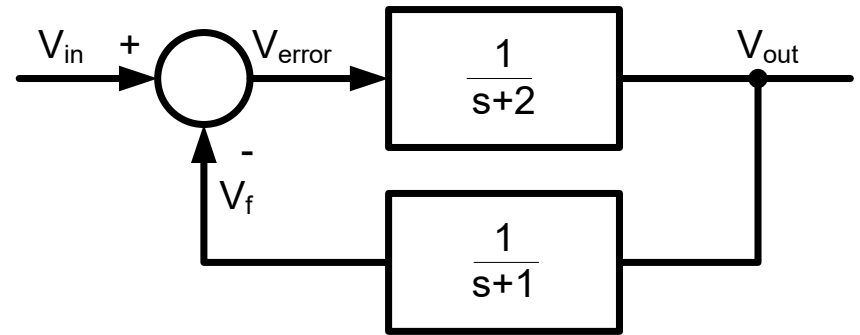
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Assume system transfer function called  $T(s)$

$$\frac{V_{out}}{V_{in}} = T(s) = \frac{G(s)}{1 + G(s)H(s)}$$

$$T(s) = \frac{\frac{1}{s+2}}{1 + \frac{1}{s+2} \frac{1}{s+1}} = \frac{\frac{1}{s+2}}{\frac{(s+2)(s+1)+1}{(s+2)(s+1)}} = \frac{s+1}{1+(s+2)(s+1)}$$

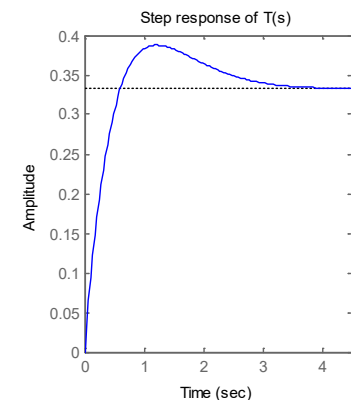
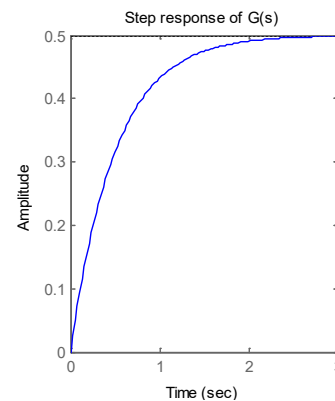
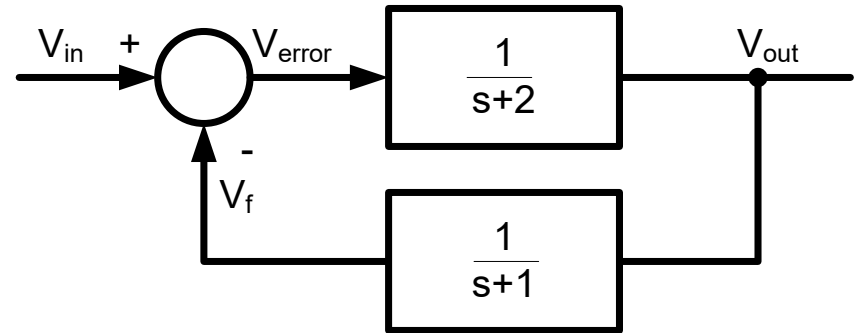
$$T(s) = \frac{s+1}{s^2 + 3s + 3}$$



# Howework #1

## □ Question

- Use matlab to calculate the system transfer function  $T(s)$ .
  - You will use matlab function
    - TF
    - FEEDBACK
- Use matlab to plot the step response of  $G(s)$  and  $T(s)$ .



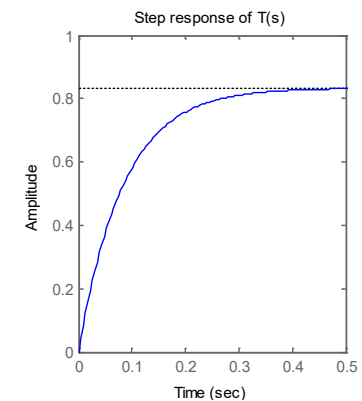
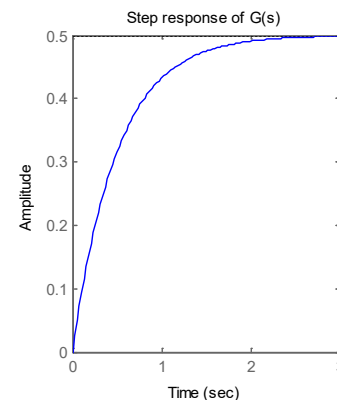
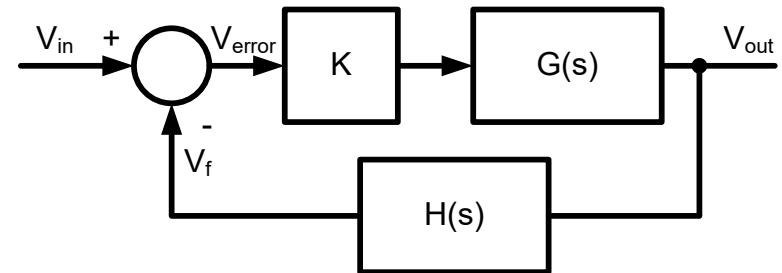
# Homework #2

## Question

### Assume

$$G(s) = \frac{1}{s+2}, \quad H(s) = 1, \quad K = 10$$

- Calculate  $T(s)$  with matlab.
- Plot step response of  $G(s)$  and  $T(s)$ .
  - What is the steady state value of  $T(s)$  in step response plot?
  - If  $K = 100$ , what is the new steady state value of  $T(s)$ ?
    - What if the function of  $K$ ?



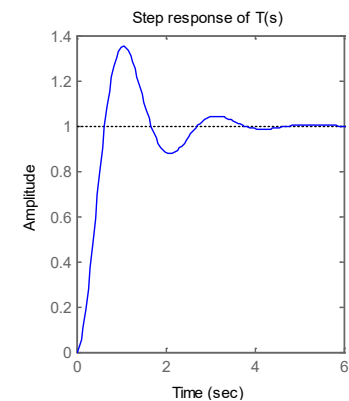
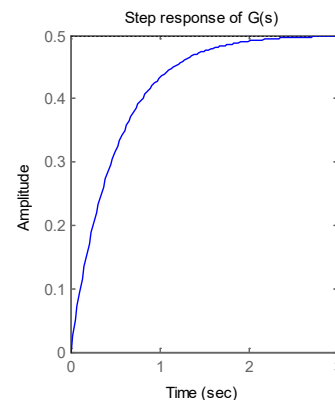
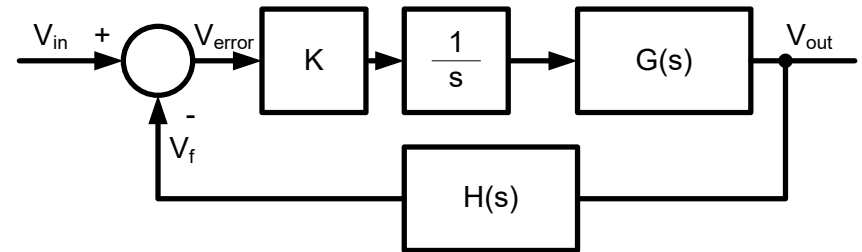
# Howework #3

## Question

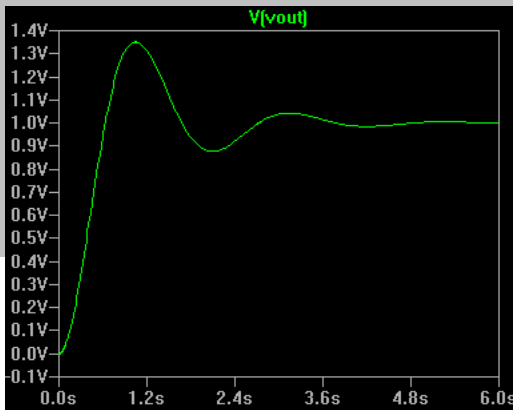
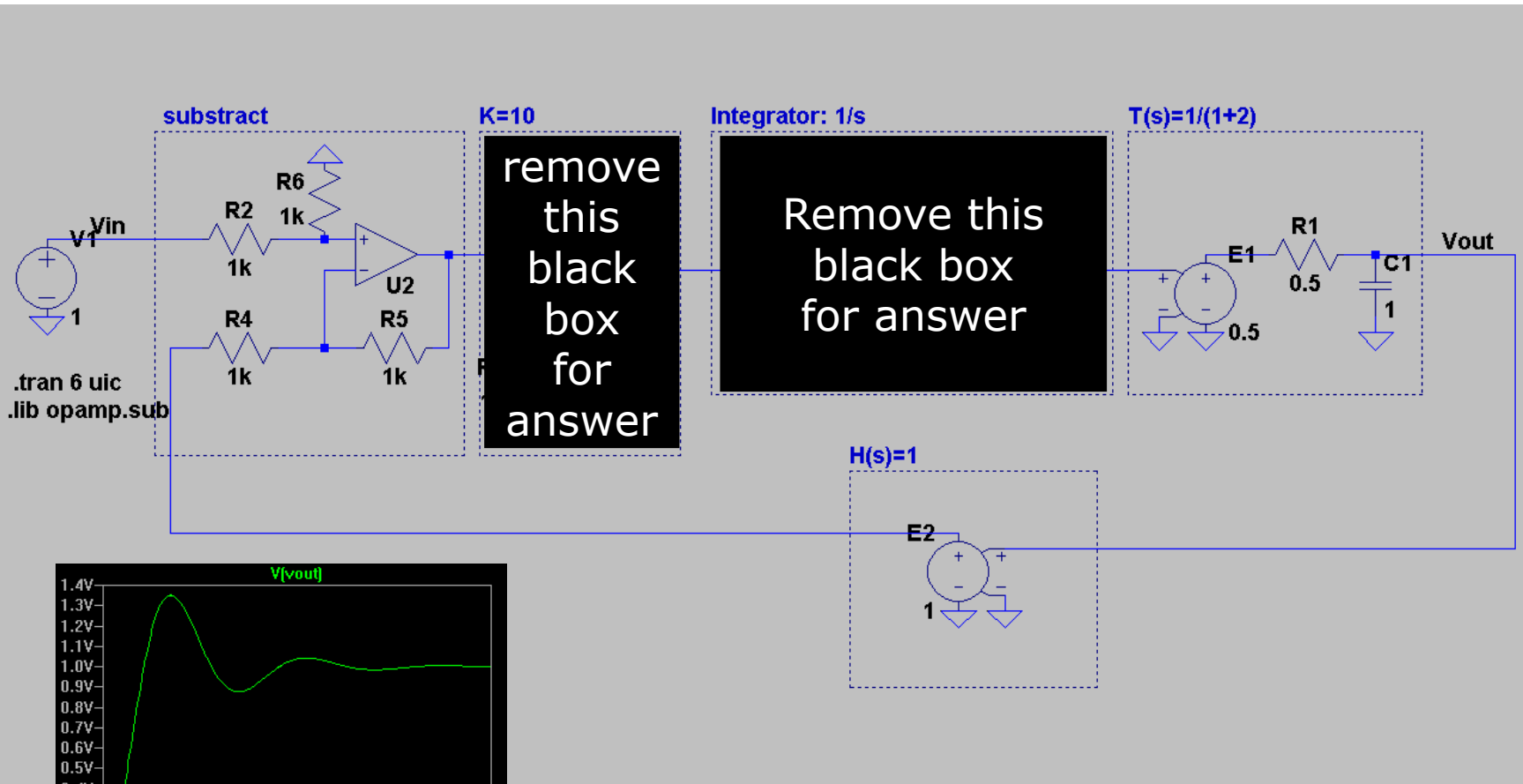
### Assume

$$G(s) = \frac{1}{s+2}, \quad H(s) = 1, \quad K = 10$$

- Calculate  $T(s)$  with matlab.
- Plot step response of  $G(s)$  and  $T(s)$ .
  - What is the steady state value of  $T(s)$  in step response plot?
  - Change  $K$  to observe  $T(s)$  step response.
    - What if the function of  $s^{-1}$  in this system?
- In next slide, design op-amp compensation network in LTspice for this circuit. Use `ans_blank.asc` as template.



# Homework #3





# Control – 1<sup>st</sup>-order and 2<sup>nd</sup>-order system response

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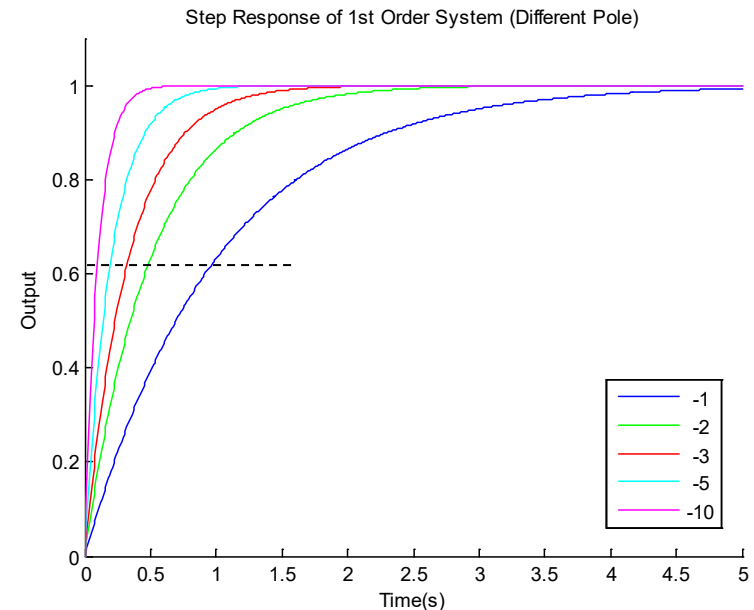
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# 1<sup>st</sup>-order System Step Response

- Assume 1<sup>st</sup>-order system as

$$G(s) = \frac{p}{s + p}$$

- Pole is defined as the root of system denominator = 0
  - i.e.  $s + p = 0 \rightarrow s = -p$
- Observation
  - A less negative pole gives a slower system.
  - A positive pole gives an unstable system.
  - System Time-Constant =  $1/p$ .
    - Time to achieve ~63% of output

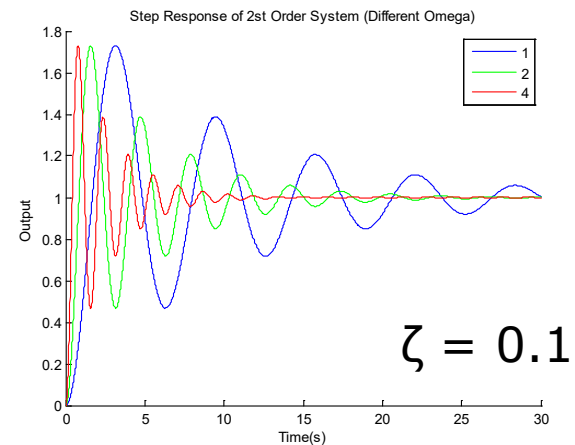
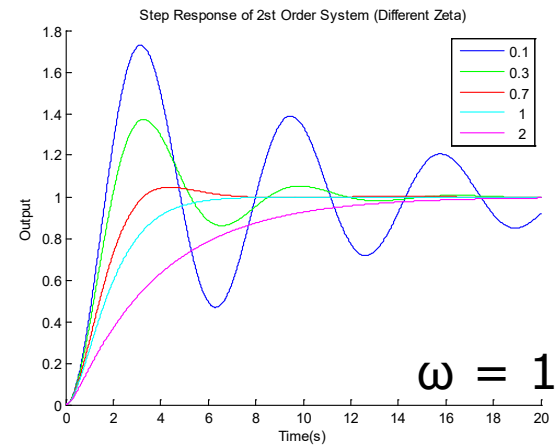


# 2<sup>nd</sup>-order System Step Response

- Assume 2<sup>nd</sup>-order system as

$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

- Pole is defined as the root of system denominator = 0
  - i.e.  $s^2 + 2\zeta\omega_n s + \omega_n^2 = 0$
- Zeta ( $\zeta$ ) Observation
  - $\zeta$  is damping ratio which affect overshoot and ringing.
  - $\zeta \sim 0.7$  can minimize overshoot but maintain system speed
- Omega ( $\omega_n$ ) Observation
  - $\omega_n$  is natural frequency which affect system speed.
  - Changing  $\omega_n$  doesn't affect overshoot or undershoot magnitude.



# System Simplification

Assume 4th - order system

$$T(s) = \frac{1}{s^4 + 42s^3 + 483s^2 + 882s + 802}$$

With the help of matlab "zpk"

$$T(s) = \frac{1}{802} \cdot \frac{401}{s^2 + 40s + 401} \cdot \frac{2}{s^2 + 2s + 2}$$

Approximation 1

$$T_1(s) = \frac{1}{802} \frac{401}{s^2 + 40s + 401}$$

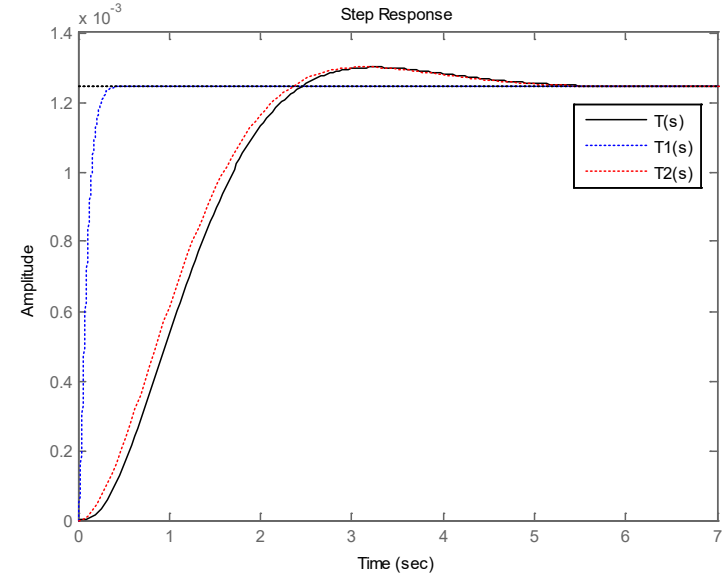
$$\omega_{n1} = \sqrt{401} \approx 20, \zeta_1 = 0.999$$

Approximation 2

$$T_2(s) = \frac{1}{802} \frac{2}{s^2 + 2s + 2}$$

$$\omega_{n2} = \sqrt{2} \approx 1.414, \zeta_2 = 0.707$$

As  $\omega_{n2} \ll \omega_{n1}$ ,  $T_2(s)$  can be used as approximation.



↑  
Smaller value represents slower system

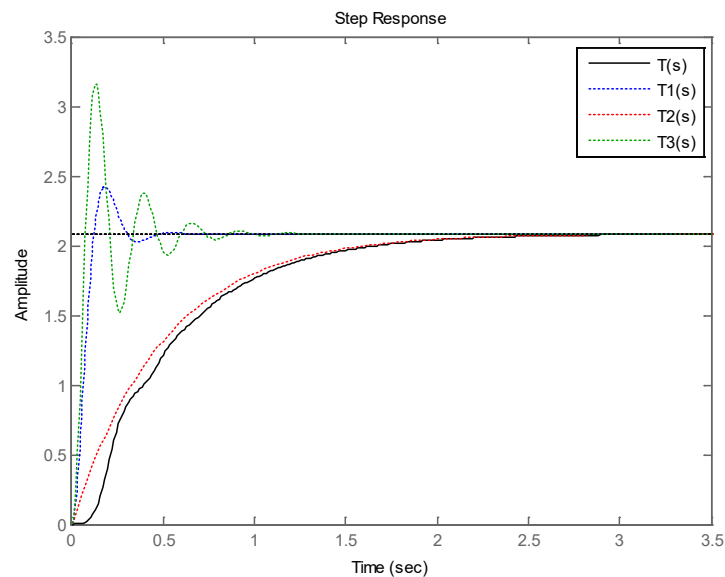
# Homework 1

- Simplify the 5-th order system  $T(s)$

$$T(s) = \frac{1000000}{s^5 + 32s^4 + 1260s^3 + 18400s^2 + 272000s + 480000}$$

- Help

- Use matlab "tf" to create the transfer functions, then use "zpk" for factorization.
- What is the approximate time-constant of the 5-th order system  $T(s)$ ?
- What is the purpose of System Simplification and the understanding of 1st and 2nd order system response?



# Root-Locus Method



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# Fundamental of Root Locus

Close - loop system is defined as,

$$T(s) = \frac{KG(s)}{1 + KG(s)H(s)}$$

where open - loop system is  $G(s)H(s)$

$$\text{Assume } G(s) = \frac{\text{num}_G}{\text{den}_G} \text{ and } H(s) = \frac{\text{num}_H}{\text{den}_H}$$

Poles of  $T(s)$  is the root of eqn  $1 + KG(s)H(s) = 0$

$$\therefore 1 + K \frac{\text{num}_G}{\text{den}_G} \frac{\text{num}_H}{\text{den}_H} = 0$$

$$\therefore \text{den}_G \text{den}_H + K \cdot \text{num}_G \text{num}_H = 0$$

$$\text{If } K = 0 \Rightarrow \text{den}_G \text{den}_H = 0$$

therefore, open - loop system poles are close - loop system poles at  $K = 0$

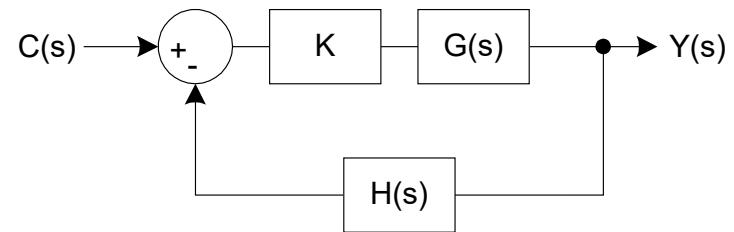
$$\text{If } K \rightarrow \infty \Rightarrow \text{num}_G \text{num}_H = \lim_{K \rightarrow \infty} \frac{-\text{den}_G \text{den}_H}{K} = 0$$

therefore, open - loop system zeros are close - loop system poles at  $K \rightarrow \infty$

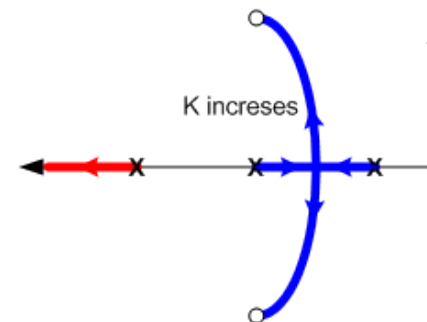
Therefore, determine the trajectories of  $1 + KG(s)H(s) = 0$  for  $K = 0$  to  $\infty$  is the locus of close - loop system poles.

Remark:

Poles of  $G(s)H(s)$  can force  $\text{den}_G=0$  or  $\text{den}_H=0$   
Zeros of  $G(s)H(s)$  can force  $\text{num}_G=0$  or  $\text{num}_H=0$



Root Locus: Trajectories of Close-Loop System Poles



X Poles of Open-Loop System, Poles of Close-Loop System @  $K = 0$   
O Zeros of Open-Loop System, Poles of Close-Loop System @  $K = \infty$

# Matlab of Root Locus

## □ Matlab Code

```
% define the open-loop system as G = (-0.5s+1)/(s^2+s)
```

```
num=[-0.5 1];
```

```
den=[1 1 0];
```

```
G = tf(num,den);
```

Open - Loop Transfer Function as

$$G(s) = \frac{-0.5s + 1}{s^2 + s}$$

```
% calculate pole and zero of close-loop systems
```

```
K = 0.647;
```

```
T = feedback(K*G,1);
```

```
[p,z]=pzmap(T);
```

```
plot(real(p),imag(p),'rd'); hold on;
```

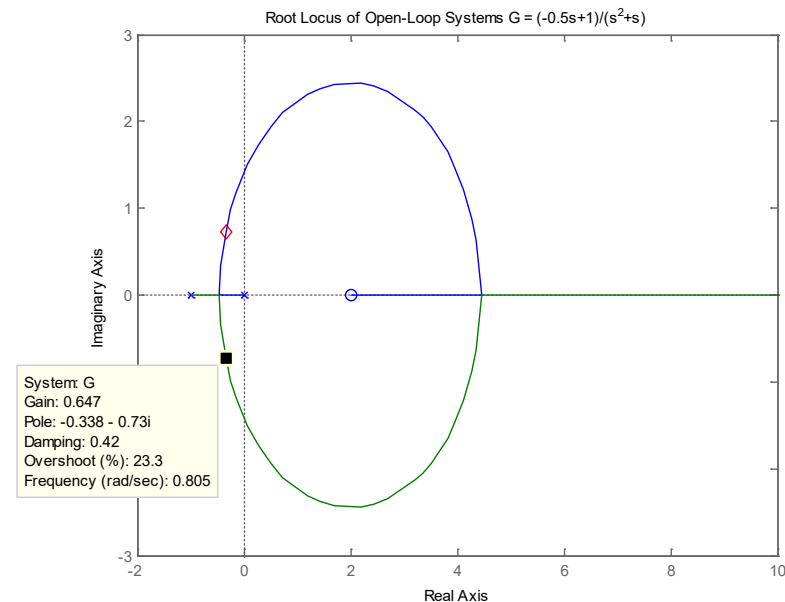
Calculate Close - Loop Transfer Function as

$$T = \frac{KG(s)}{1 + KG(s)}$$

```
% plot root-locus of open-loop systems
```

```
rlocus(G)
```

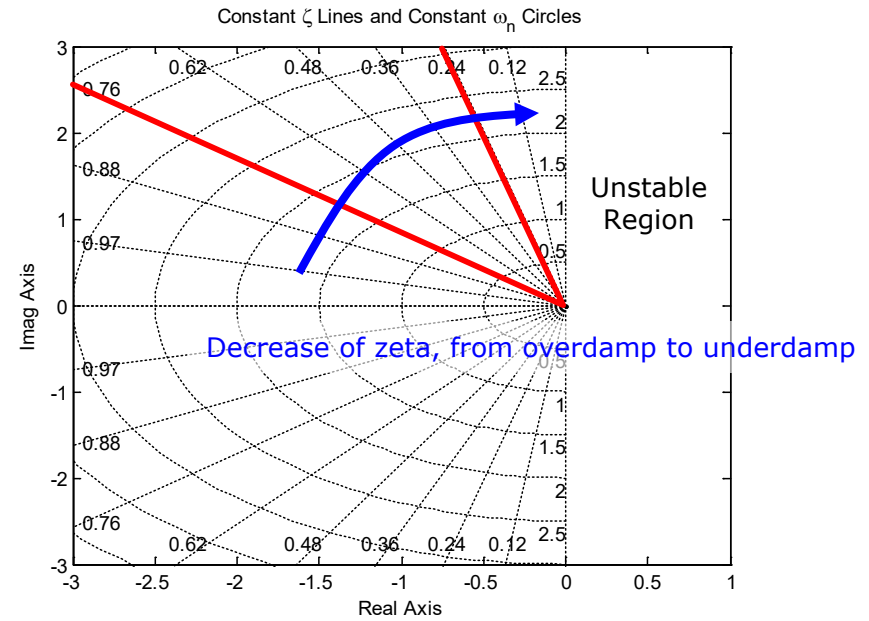
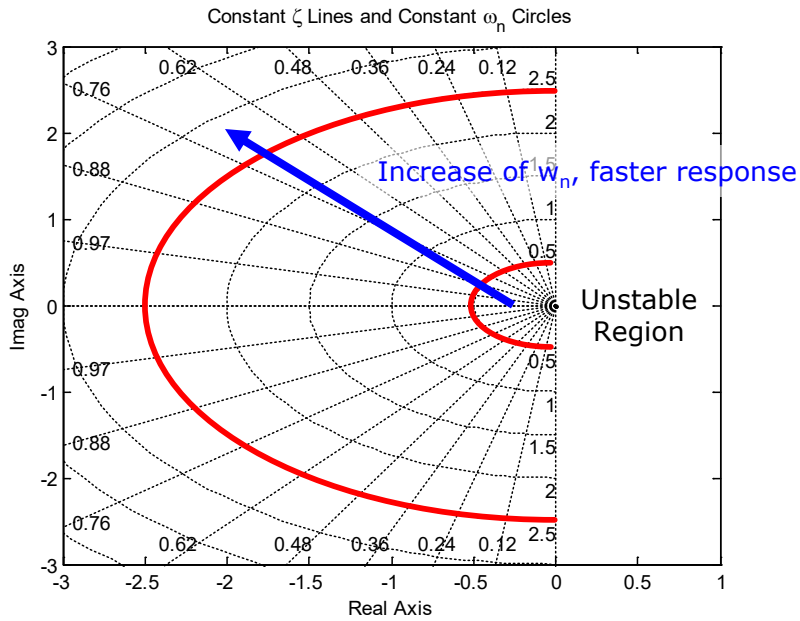
```
title('Root Locus of Open-Loop Systems G = (-0.5s+1)/(s^2+s)');
```



Prove from this matlab routine:  
rlocus plots the root locus of open-loop system,  
which represents the locus of poles of close-loop system with K from 0 to INF.



# Root Locus Plot (sgrid)



# System dynamic design criteria

$\omega_n$  is natural frequency

$\zeta$  is damping ratio

where  $\sigma = \omega_n \zeta$ ,  $\omega_d = \omega_n \sqrt{1 - \zeta^2}$

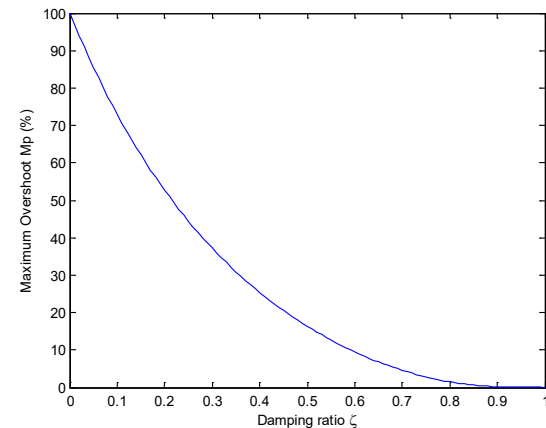
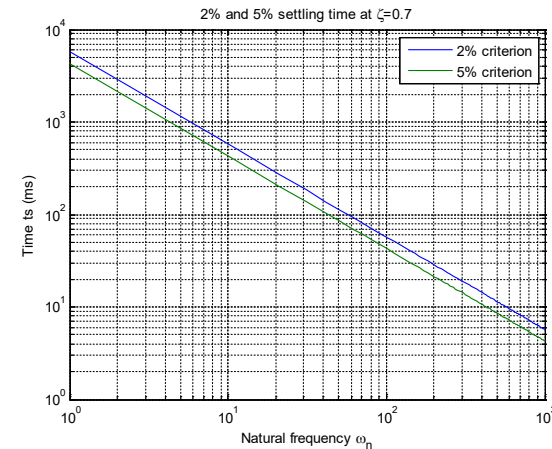
Settling time ( $t_s$ )

$$2\% \text{ criterion: } t_s = 4T = \frac{4}{\sigma} = \frac{4}{\omega_n \zeta}$$

$$5\% \text{ criterion: } t_s = 3T = \frac{3}{\sigma} = \frac{3}{\omega_n \zeta}$$

Maximum overshoot ( $M_p$ )

$$M_p = e^{-\frac{\sigma}{\omega_d} \pi} = e^{-\frac{\zeta}{\sqrt{1-\zeta^2}} \pi}$$



# Design with Root-Locus Method

## Matlab Code

```
% define the open-loop system as G(s)
```

```
z=[];
```

```
p=[0 -5 -10];
```

```
k=1;
```

```
G = zpk(z,p,k);
```

Open - Loop Transfer Function as

$$G(s) = \frac{1}{s(s+5)(s+10)}$$

```
% plot root-locus of open-loop systems
```

```
figure;
```

```
K=[0:0.2:4e3];
```

```
rlocus(G,K); sgrid;
```

```
axis([-12 2 -20 20]);
```

```
title('Constant \zeta Lines and Constant \omega_n Circles');
```

```
% form close-loop systems T(s)
```

```
K = 82.6;
```

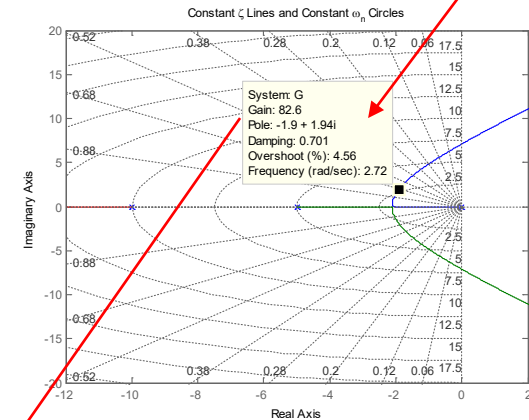
```
T = feedback(K*G,1);
```

```
% plot step response of close-loop system
```

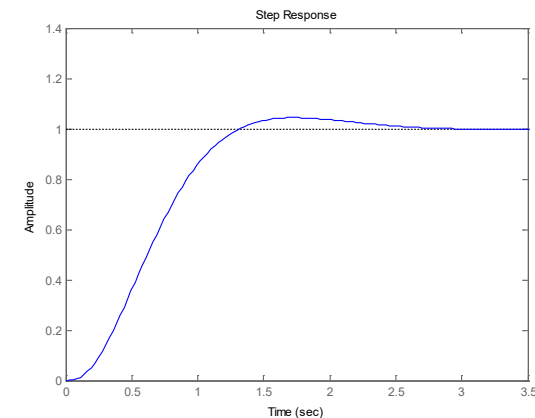
```
figure;
```

```
step(T);
```

(1) In root locus plot, put a marker and search for zeta = 0.7



(2) Gain=82.6  
for zeta = 0.7



# Design with Root-Locus Method

## (Improve response with addition zeros)

### □ Design of compensator

#### ■ Reference

- P.310 of Modern Control Engineering (5th Edition), Katsuhiko Ogata.

#### ■ Effects of the addition of poles

- Pulling the root locus to the right, tending to lower the system's relative stability and to slow down the settling of the response.

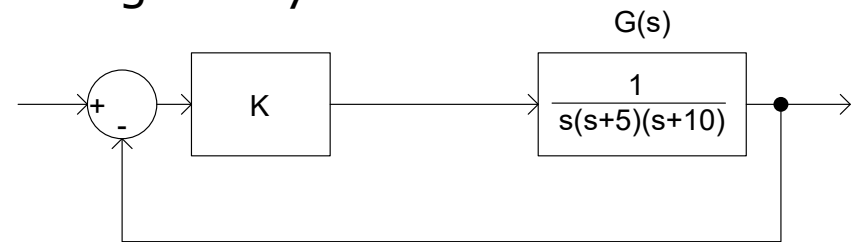
#### ■ Effects of the addition of zeros

- Pulling the root locus to the left, tending to make the system more stable and to speed up the settling of the response.

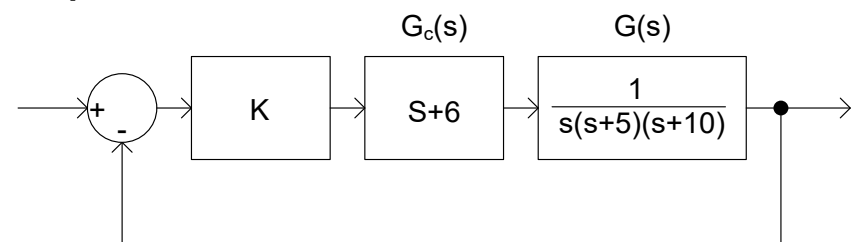
### □ Matlab example

- Based on previous design, we add a compensator with zero = -6.

### Original System

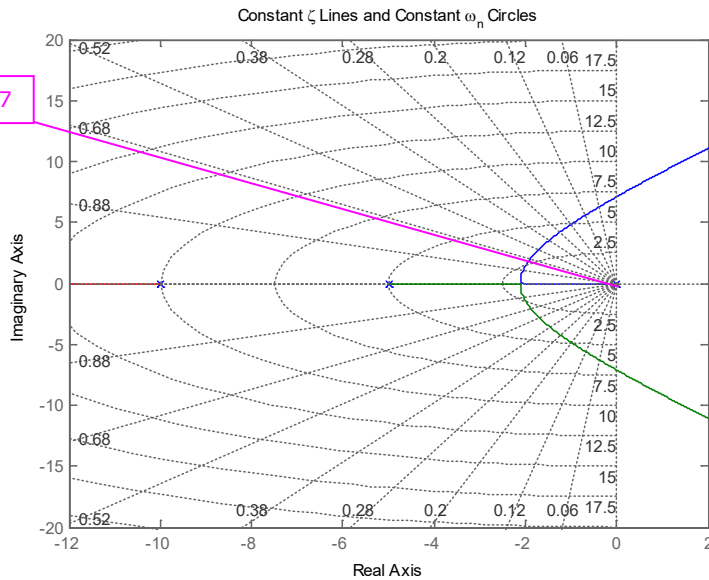


### System with addition zero



# Design with Root-Locus Method (Improve response with addition zeros)

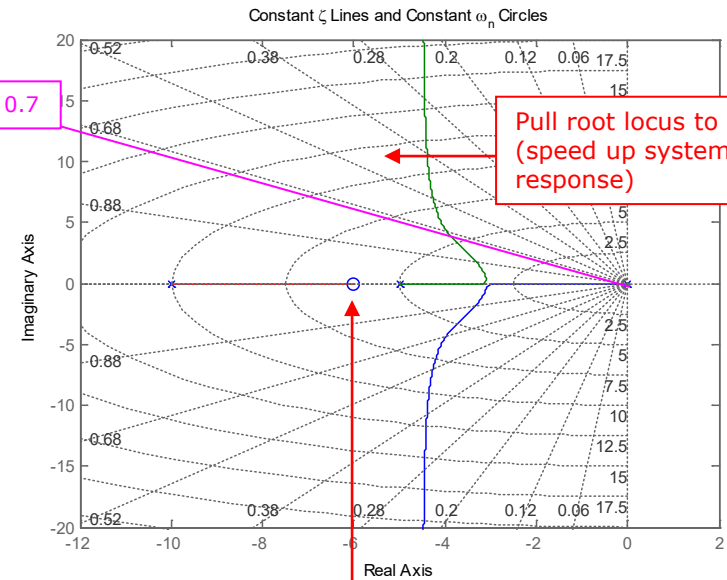
## Original System



### Observation

- Almost double  $\omega_n$  at zeta = 0.7
- Close-loop stable for all K.

## System with addition zero



# Design with Root-Locus Method

## (Improve response with addition zeros)

### Matlab Code

```

% define the open-loop system as G(s)
z=[];
p=[0 -5 -10];
k=1;
G = zpk(z,p,k);

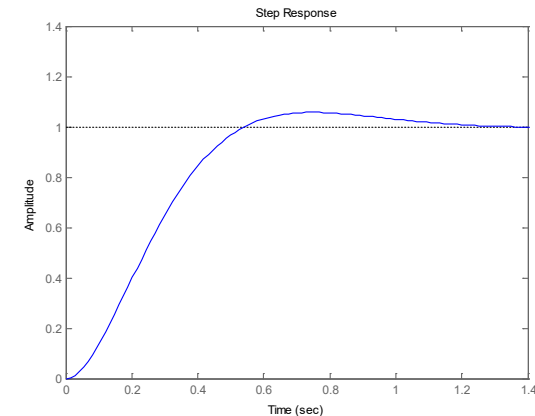
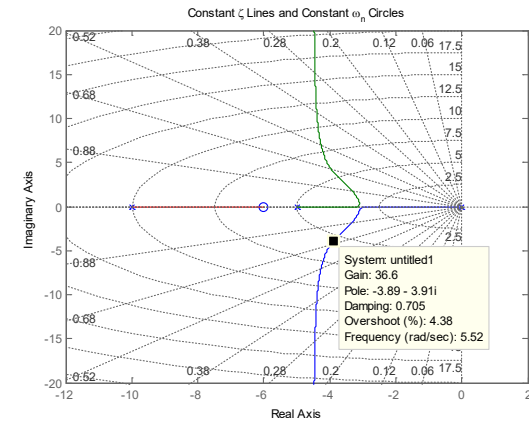
% compensator transfer function Gc(s)
num=[1 6];
den=[1];
Gc = tf(num,den);

% plot root-locus of open-loop systems Gc(s)G(s)
figure;
K=[0:0.2:1e3];
rlocus(Gc*G,K); sgrid;
axis([-12 2 -20 20]);
title('Constant \zeta Lines and Constant \omega_n Circles');

% form close-loop systems T(s)
K = 36.6;
T = feedback(K*Gc*G,1);

% plot step response of close-loop system
figure;
step(T);

```

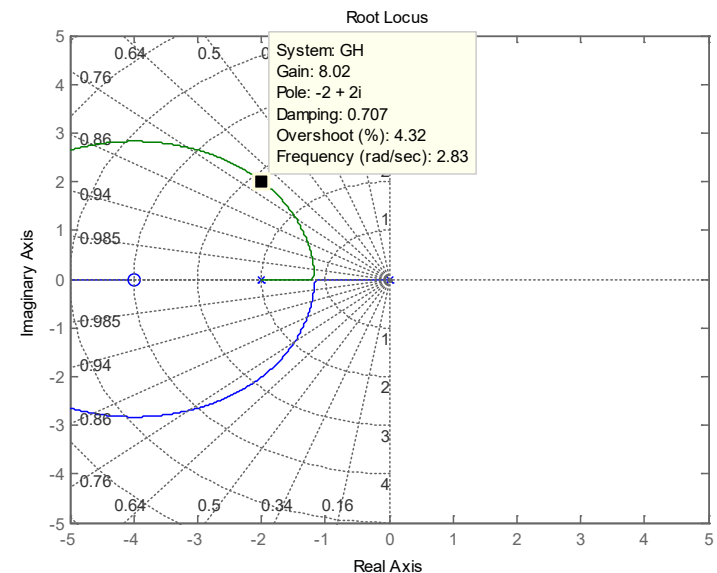
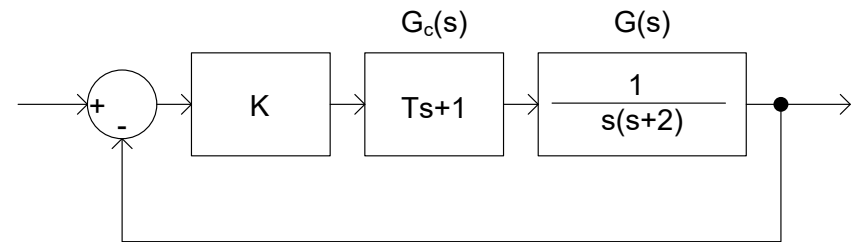


# Exercise #1

## □ Exercise #1

- Use matlab to determine the gain  $K$  and time constant  $T$  of the controller  $G_c(s)$  such that the closed-loop poles are located at  $s = -2 \pm j2$ .

- Hint:  $T$  can be determined by trial and error method in matlab.



# Rules for Constructing Root Locus

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10-19-2012

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# Rules for Constructing Root Loci

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## □ Reference

- P.283-287, "Modern Control Engineering", Fifth Edition, Katsuhiko Ogata
- The construction rules in this ppt follows the Ogata textbook.

Root Locus is the root trajectory of Characteristic Equation

$$den_G den_H + K \cdot num_G num_H = 0$$

or

$$1 + K \cdot G(s)H(s) = 0$$

or represented in general form

$$B(s) + K \cdot A(s) = 0$$

where

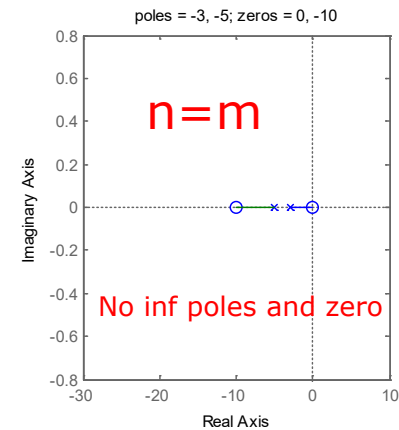
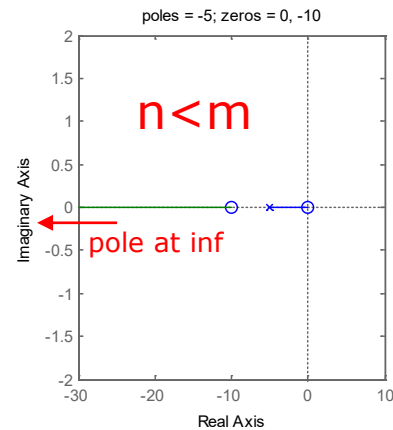
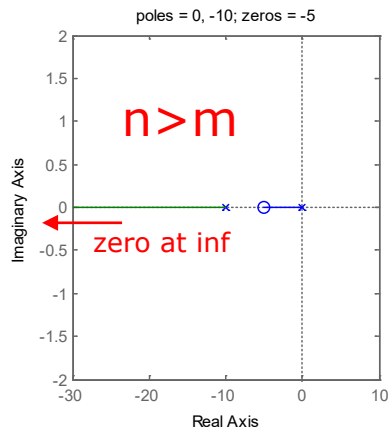
roots of  $A(s) = 0$  are open - loop system zeros

roots of  $B(s) = 0$  are open - loop system poles

Therefore, root locus can apply for any system which re - write to this general form.

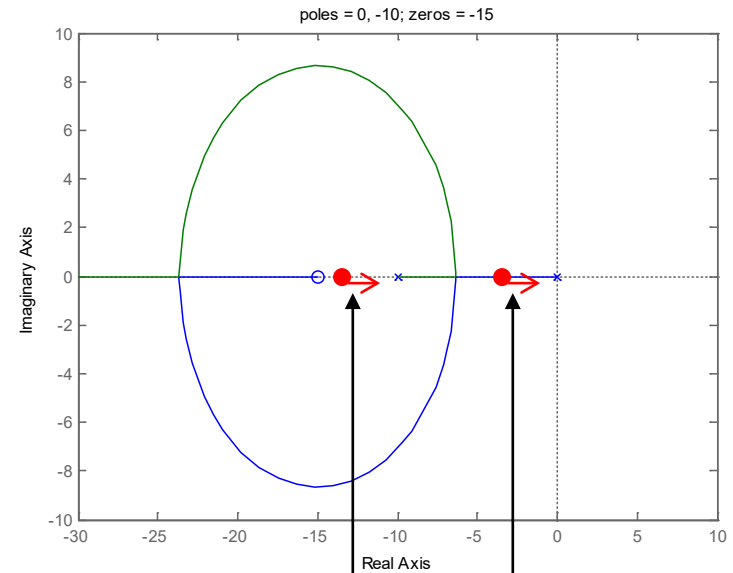
# Rule #1

- Locate the poles and zeros of  $G(s)H(s)$  on the  $s$  plane.
  - Root-locus branches start from open-loop poles and terminate at zeros (finite zeros or zeros at infinity)
  - Assume
    - Number of poles =  $n$
    - Number of zeros =  $m$ 
      - If  $n > m$ , then system has  $n - m$  infinity zeros
      - If  $n < m$ , then system has  $m - n$  infinity poles
      - If  $n = m$ , then system has no infinity poles or zeros.



# Rule #2

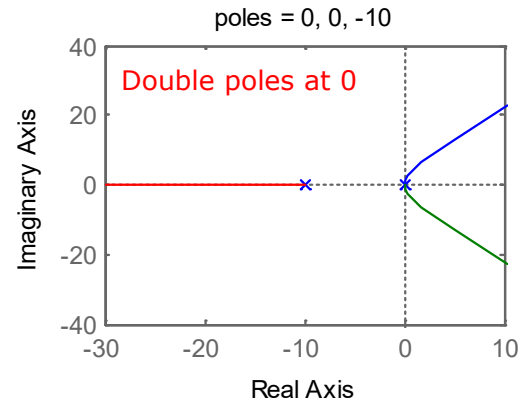
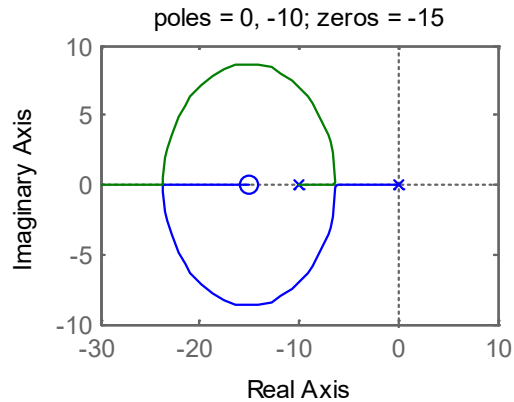
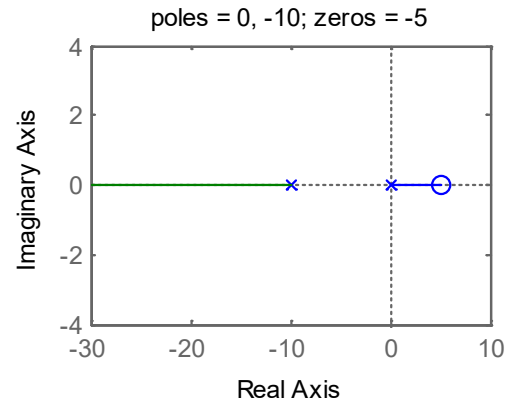
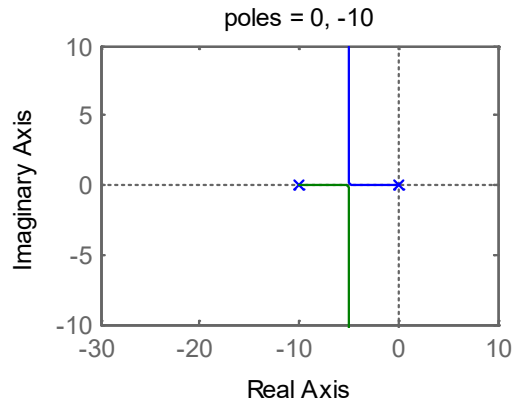
- Determine the root loci on the real axis
  - Put a test point on the real axis,
    - If the total poles and zeros to the right of this test point is odd, then this point lies on the root locus, otherwise, point doesn't lie on the root locus.



poles+zeros = 2, even, no root locus

poles+zeros = 1, odd, lie on root locus

# Rule #2 (examples)



# Rule #3

## □ Determine the asymptotes of root loci

$$\text{Angles of asymptotes} = \frac{\pm 180^\circ(2k+1)}{n-m} \quad (k=0,1,2,\dots)$$

where

$n$  = number of finite poles of  $G(s)H(s)$

$m$  = number of finite zeros of  $G(s)H(s)$

$$\text{If open-loop system is } G(s)H(s) = \frac{\prod (s+z_n)}{\prod (s+p_n)}$$

Intersection of asymptotes

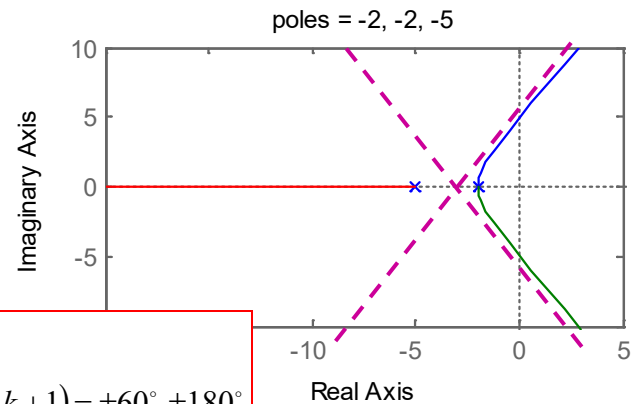
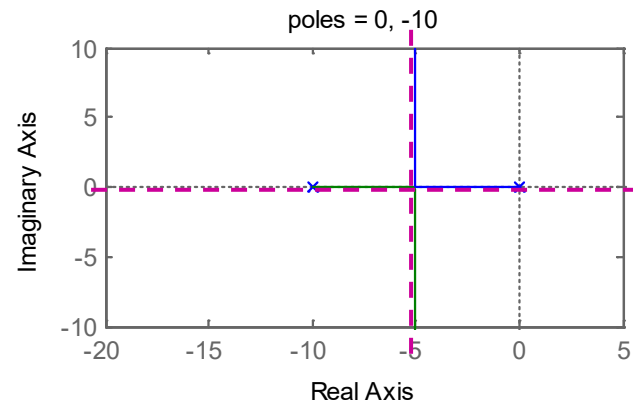
$$s_{\text{intersection}} = -\frac{\sum p_n - \sum z_n}{n-m}$$

Caution  $p_n$  and  $z_n$  not poles and zeros  
For example, if poles are -1, -2,  $p_1=1$ ,  $p_2=2$   
 $(s+p_1)(s+p_2)$

$$n=2, m=0$$

$$\text{angles of asymptotes} = \frac{\pm 180^\circ(2k+1)}{2} = \pm 90^\circ(2k+1) = \pm 90^\circ, \pm 180^\circ$$

$$s_{\text{intersection}} = -\frac{\sum p_n - \sum z_n}{n-m} = -\frac{(0+10)-0}{2} = -5$$



$$n=3, m=0$$

$$\text{angles of asymptotes} = \frac{\pm 180^\circ(2k+1)}{3} = \pm 60^\circ(2k+1) = \pm 60^\circ, \pm 180^\circ$$

$$s_{\text{intersection}} = -\frac{\sum p - \sum z}{n-m} = -\frac{(2+2+5)-0}{3} = -3$$

# Rule #4

---

## □ Find the breakaway and break-in points

Identify the characteristic equation in this format

$$B(s) + K \cdot A(s) = 0$$

Breakaway or Break - in points are the roots of

$$\frac{dK}{ds} = -\frac{A(s)\frac{dB(s)}{ds} - B(s)\frac{dA(s)}{ds}}{(A(s))^2} = 0$$

$$\therefore A(s)\frac{dB(s)}{ds} - B(s)\frac{dA(s)}{ds} = 0$$

A actual breakaway or break - in point can obtain  $K$  as a positive number

by substitute that  $s$  (root of  $\frac{dK}{ds} = 0$ ) into characteristic equation.

# Rule #4

Assume Open - Loop system

$$G(s)H(s) = \frac{s+15}{s(s+10)} = \frac{s+15}{s^2+10s}$$

∴ Characteristic equation is

$$1 + K \cdot G(s)H(s) = 0 \Rightarrow 1 + K \frac{s+15}{s^2+10s} = 0$$

$$\therefore (s^2+10s) + K(s+15) = 0$$

where  $A(s) = s+15, B(s) = s^2+10s$

Breakaway or break - in points

$$\text{By } \frac{dK}{ds} = -\frac{A(s)\frac{dB(s)}{ds} - B(s)\frac{dA(s)}{ds}}{(A(s))^2} = 0$$

$$\Rightarrow A(s)\frac{dB(s)}{ds} - B(s)\frac{dA(s)}{ds} = 0$$

$$\Rightarrow (s+15)\frac{d(s^2+10s)}{ds} - (s^2+10s)\frac{d(s+15)}{ds} = 0$$

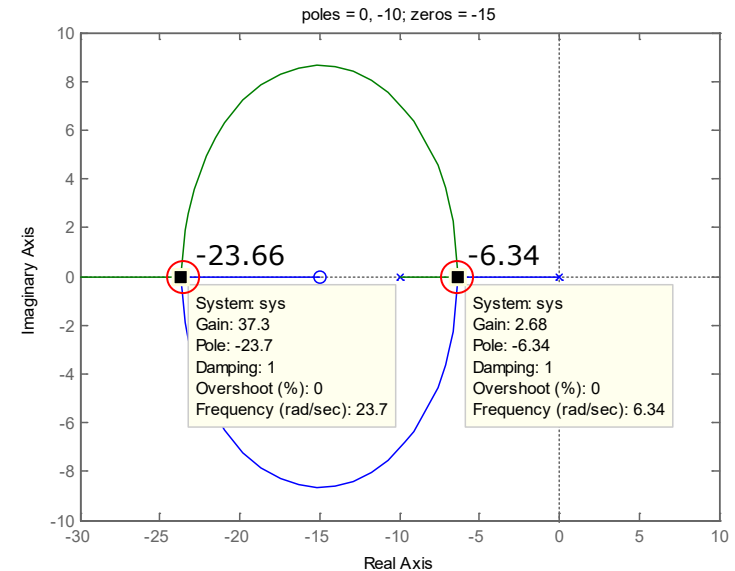
$$\Rightarrow (s+15)(2s+10) - (s^2+10s)(1) = 0$$

$$\Rightarrow s^2 + 30s + 150 = 0$$

$$\therefore s = -6.34 \text{ or } s = -23.66$$

$$\text{Put } s = -6.34 \text{ into } (s^2+10s) + K(s+15) = 0 \Rightarrow K = 2.68$$

$$\text{Put } s = -23.66 \text{ into } (s^2+10s) + K(s+15) = 0 \Rightarrow K = 37.32$$



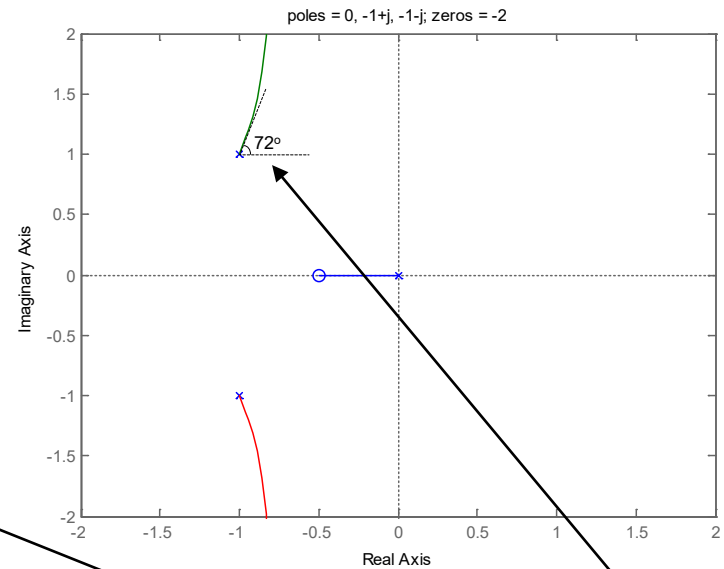
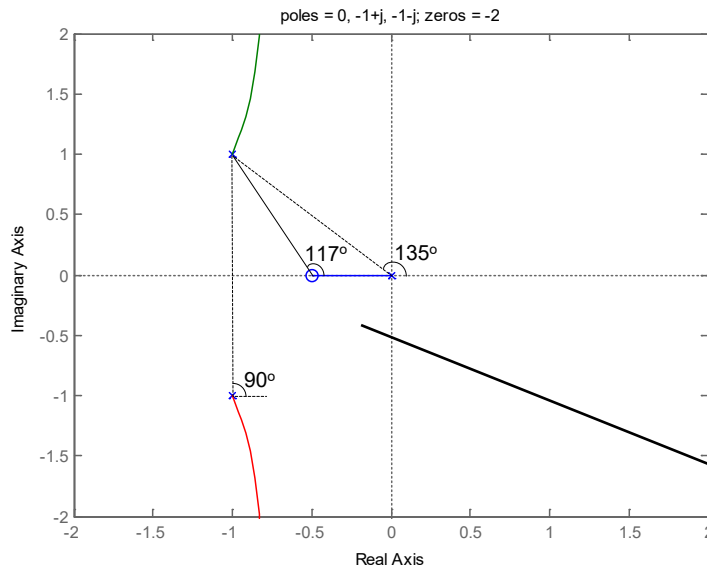
# Rule #5

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- Determine the angle of departure (angle of arrival) of the root locus from a complex pole (at a complex zero)
  - Angle of departure from a complex pole =  $180^\circ$ 
    - (sum of the angles of vectors to a complex pole in question from other poles)
    - + (sum of the angles of vectors to a complex pole in question from zeros)
  - Angle of arrival at a complex zero =  $180^\circ$ 
    - (sum of the angles of vectors to a complex zero in question from other zeros)
    - + (sum of the angles of vectors to a complex zero in question from poles)

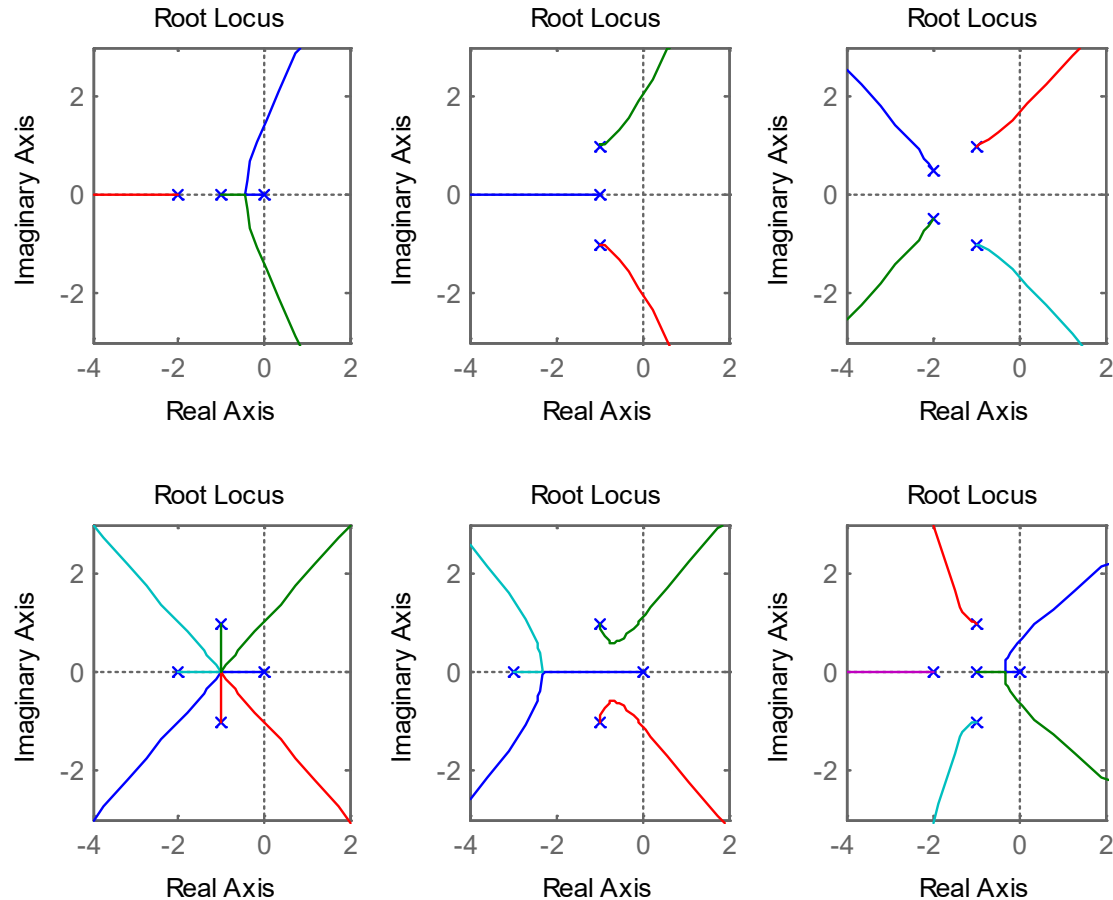


# Rule #5 (example)

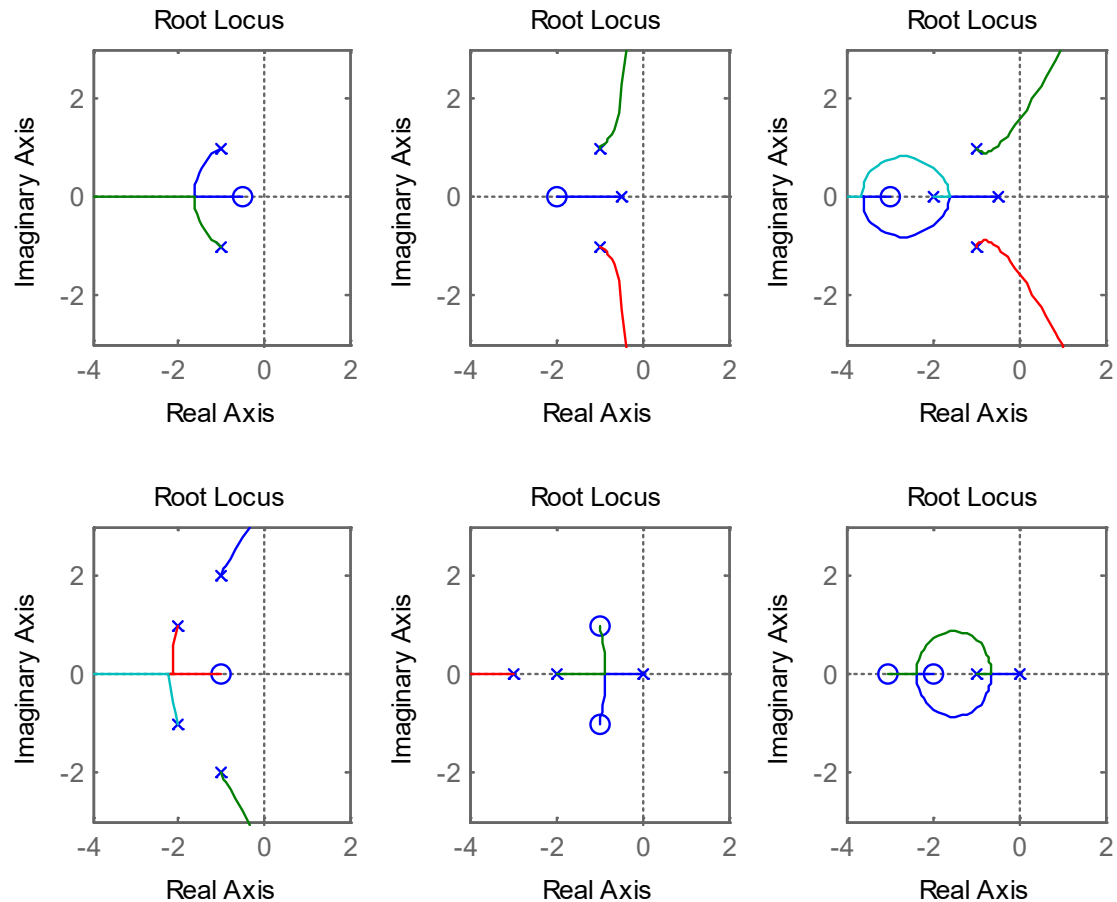


Angle of departure from a complex pole =  $180^\circ - (135^\circ + 90^\circ) + (117^\circ) = 72^\circ$

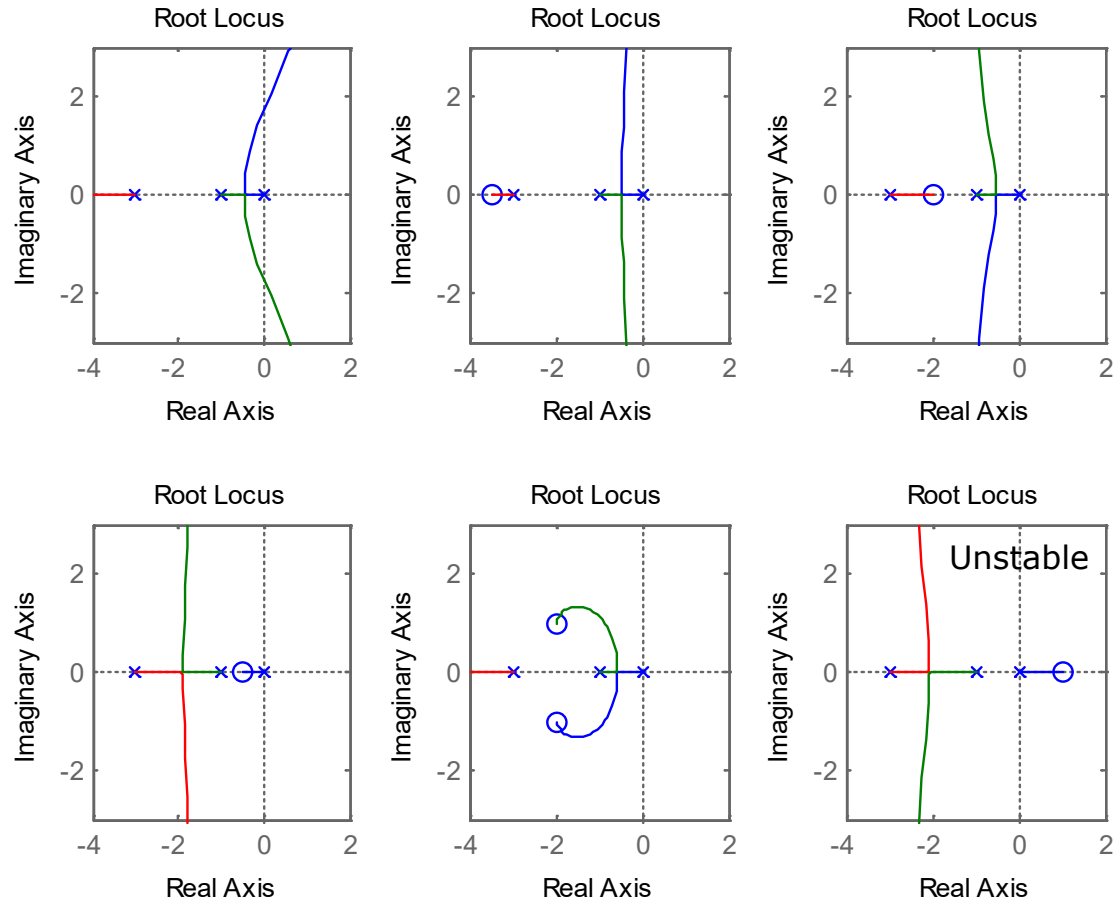
# Open-Loop Pole-Zero Configurations and the Corresponding Root Loci



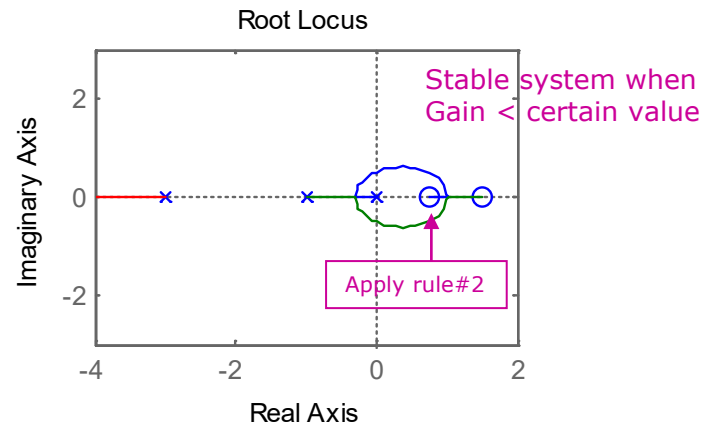
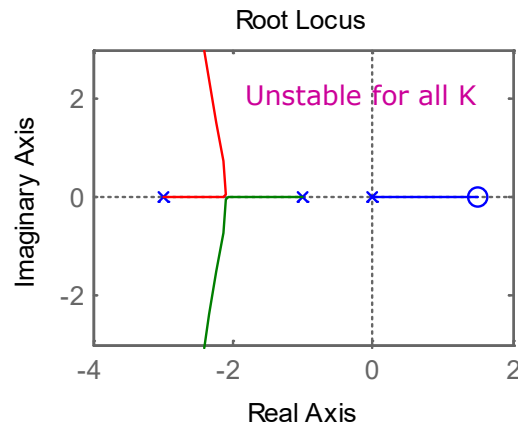
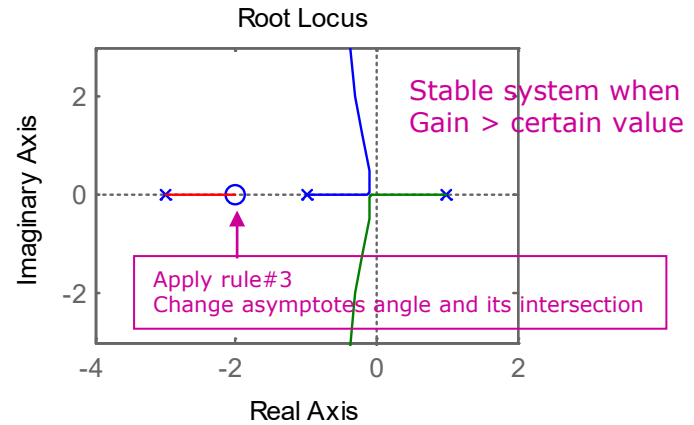
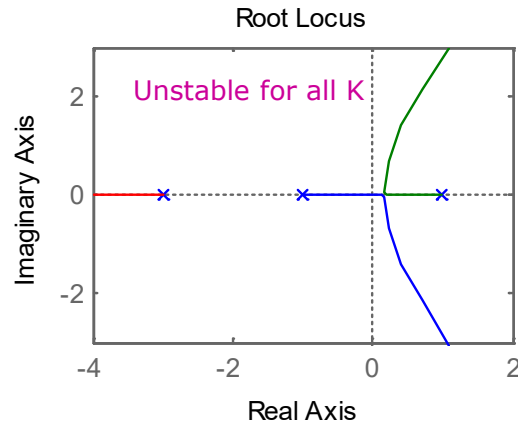
# Open-Loop Pole-Zero Configurations and the Corresponding Root Loci



# Example of compensating a system with addition zeros



# Example of compensating an unstable system



# Frequency Response Method - Bode Plot

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10-22-2012

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# Bode Plot

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## □ Bode Plot

- Presenting frequency-response characteristics in graphical forms.
- Plot of Logarithm of the magnitude of a sinusoidal transfer function
- Plot of phase angle
- Against the frequency on a logarithmic scale.

System transfer function  $G(s)$

Substitute  $s = j\omega = j2\pi f$

$G(s)$  can be expressed as

$$G(j\omega) = |G(j\omega)| \angle G(j\omega)$$

Logarithm Magnitude

$$|G(j\omega)|_{dB} = 20 \log(|G(j\omega)|)$$

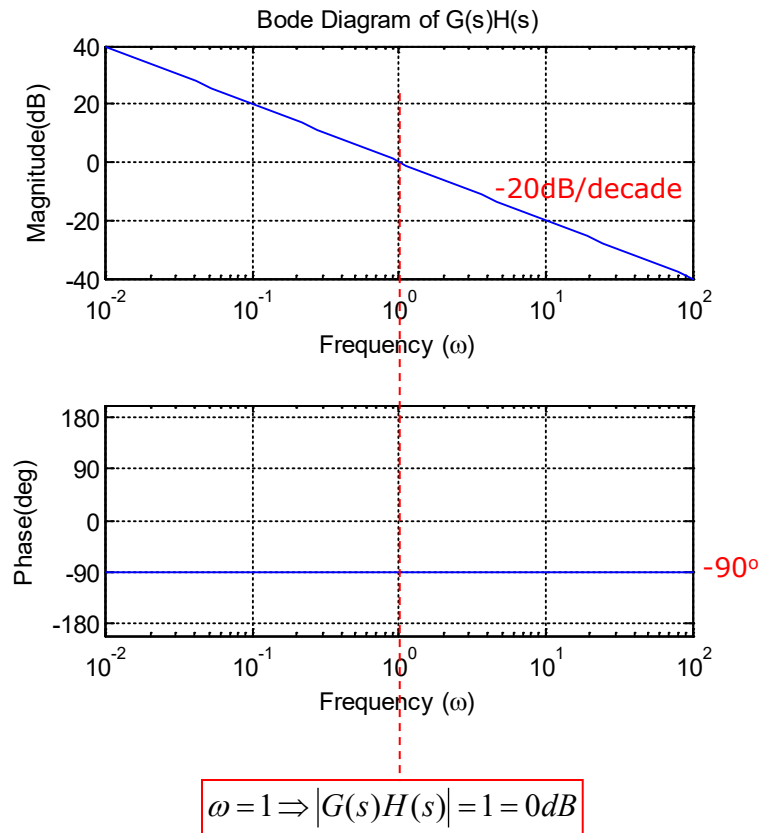
# Typical Bode Plot, $GH=1/s$

Bode Plot of

$$G(s)H(s) = \frac{1}{s}$$

Substitute  $s = j\omega$

$$G(j\omega)H(j\omega) = \frac{1}{j\omega}$$





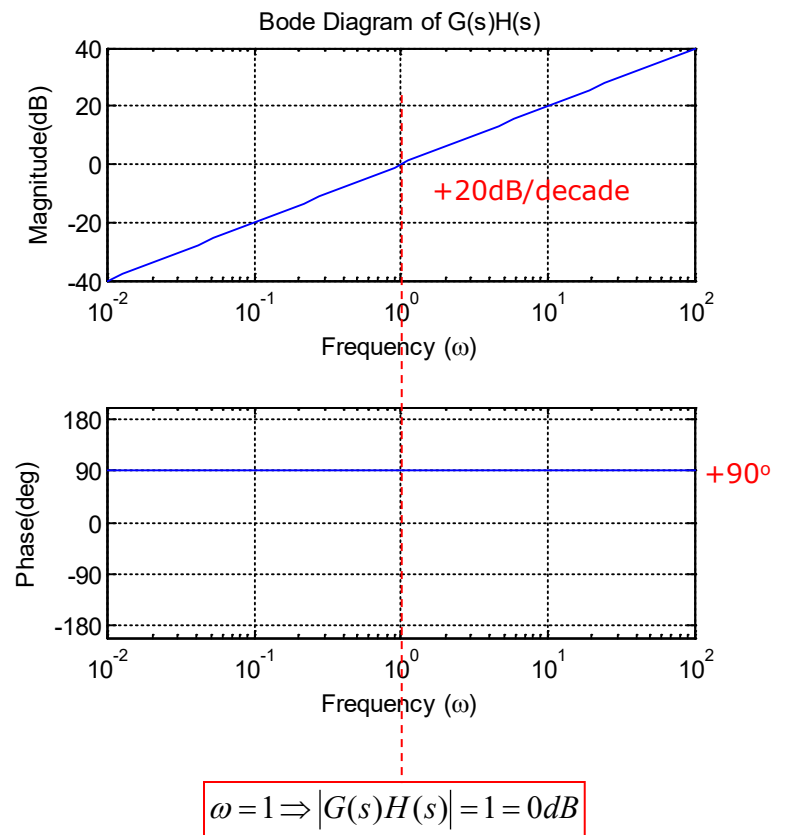
# Typical Bode Plot, $GH=s$

Bode Plot of

$$G(s)H(s) = s$$

Substitute  $s = j\omega$

$$G(j\omega)H(j\omega) = j\omega$$



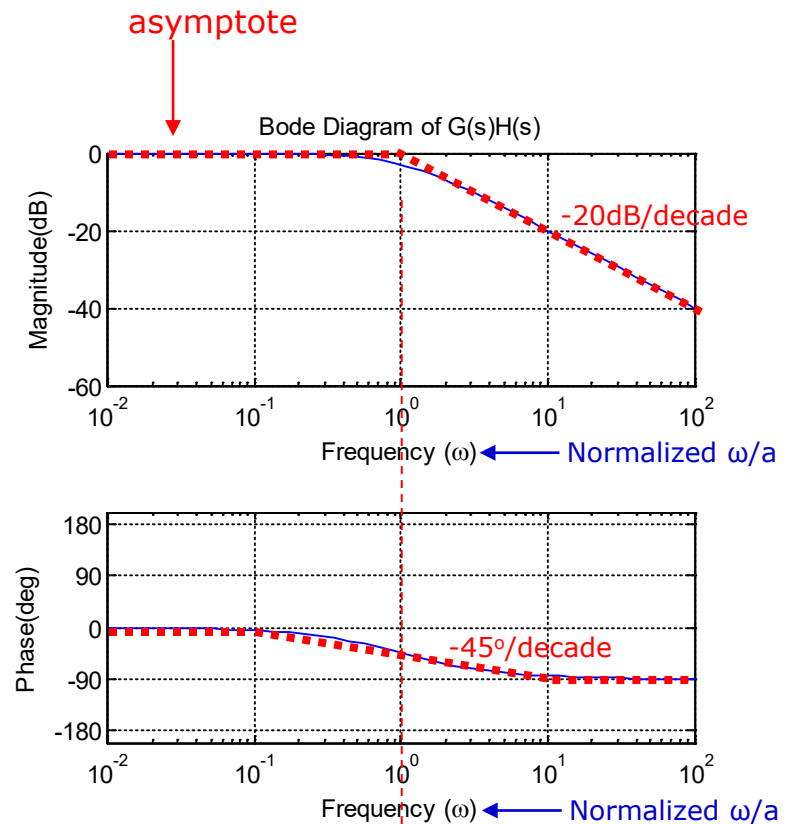
# Typical Bode Plot, $GH=a/(s+a)$

Bode Plot of

$$G(s)H(s) = \frac{a}{s+a}$$

Substitute  $s = j\omega$

$$G(j\omega)H(j\omega) = \frac{a}{j\omega+a}$$



$$\omega = a$$

$$\Rightarrow |G(s)H(s)| = 1 = 0dB$$

$$\Rightarrow \angle G(s)H(s) = 45^\circ$$

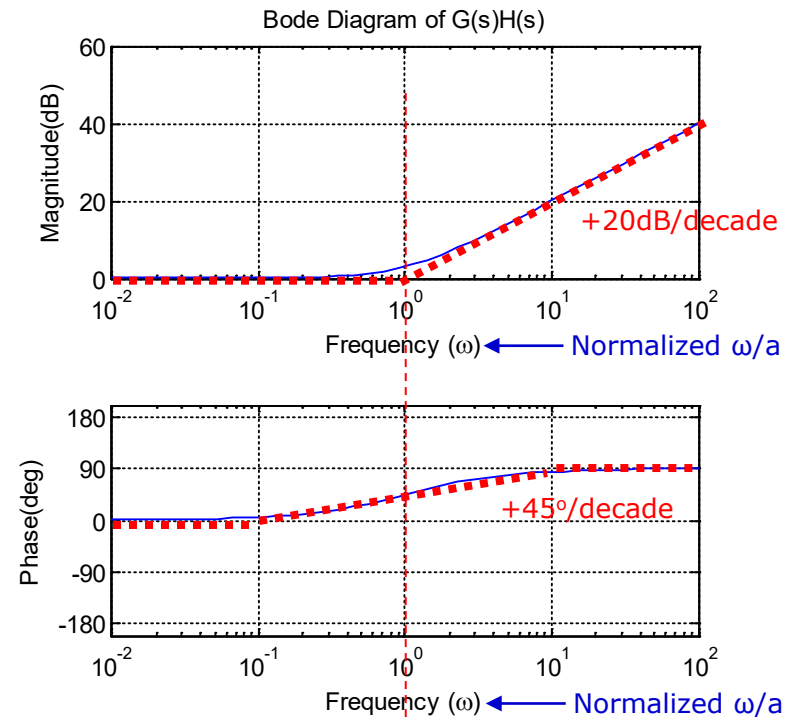
# Typical Bode Plot, $GH=(s+a)/a$

Bode Plot of

$$G(s)H(s) = \frac{s+a}{a}$$

Substitute  $s = j\omega$

$$G(j\omega)H(j\omega) = \frac{j\omega+a}{a}$$



$$\omega = a$$

$$\Rightarrow |G(s)H(s)| = 1 = 0dB$$

$$\Rightarrow \angle G(s)H(s) = 45^\circ$$

# Typical Bode Plot, Second-order

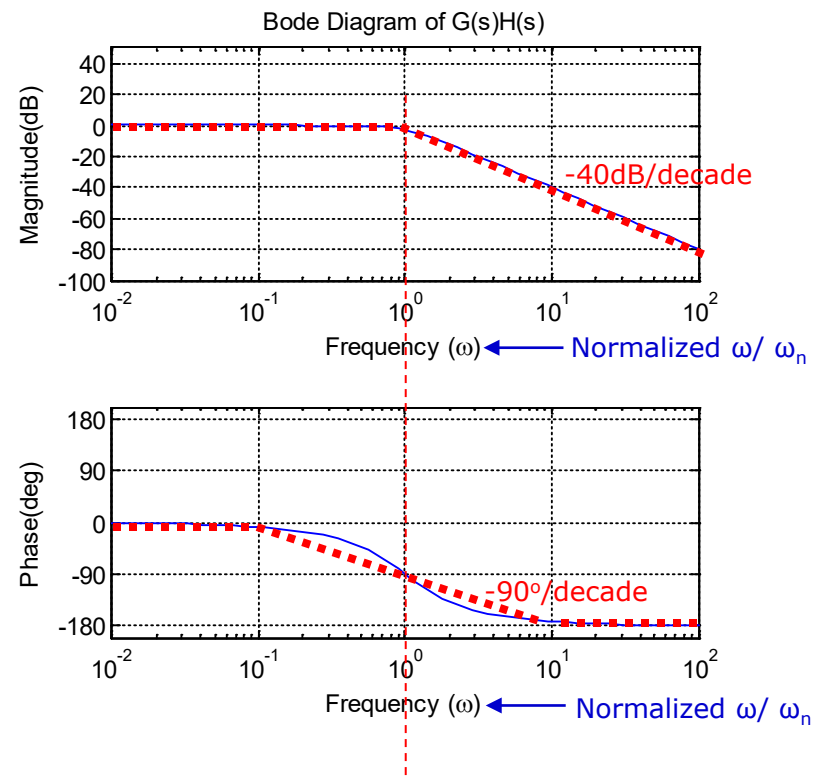
This example:  $\omega_n=1, \zeta=0.7$

Bode Plot of

$$G(s)H(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

Substitute  $s = j\omega$

$$G(j\omega)H(j\omega) = \frac{\omega_n^2}{(j\omega)^2 + 2\zeta\omega_n(j\omega) + \omega_n^2}$$



$$\omega = \omega_n$$

$$\Rightarrow \angle G(s)H(s) = 90^\circ$$

# Example 1: Construct of Bode Plot

Plot the Bode Plot of  $G(s)H(s)$

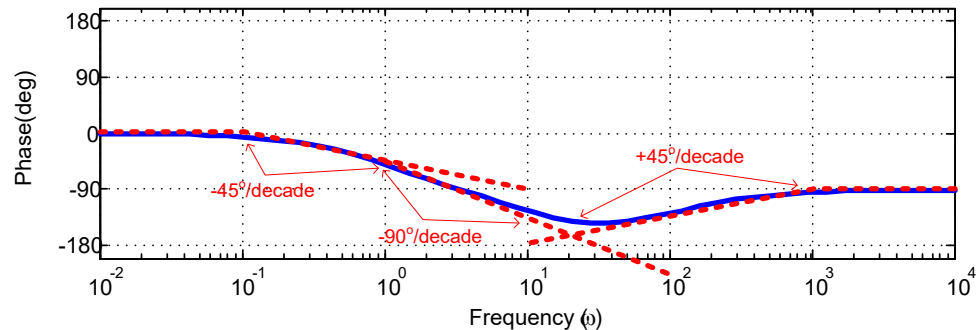
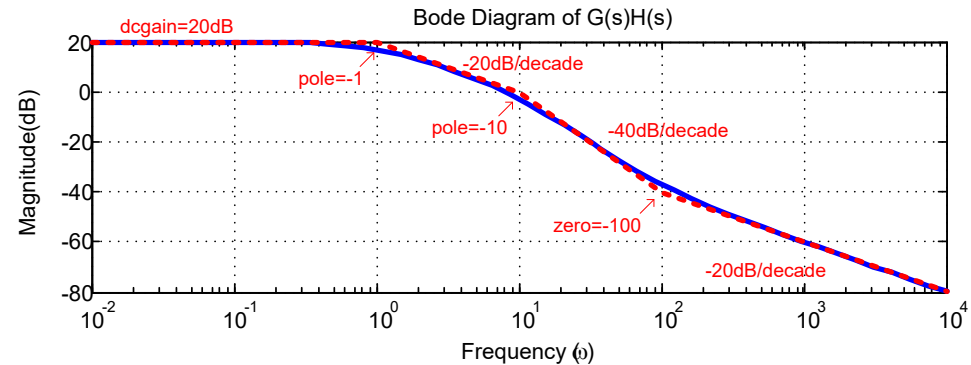
$$G(s)H(s) = \frac{s+100}{(s+1)(s+10)}$$

Re - write the  $G(s)H(s)$  into standard form

$$G(s)H(s) = \frac{100}{1 \cdot 10} \frac{1}{s+1} \frac{10}{s+10} \frac{s+100}{100}$$

$$G(s)H(s) = 10 \cdot \frac{1}{s+1} \frac{10}{s+10} \frac{s+100}{100}$$

dcgain = 10  
 dcgain(dB) =  $20 \cdot \log(\text{dcgain}) = 20\text{dB}$



# Example 2: Construct of Bode Plot

Plot the Bode Plot of  $G(s)H(s)$

$$G(s)H(s) = \frac{100000}{s(s^2 + 100s + 5000)}$$

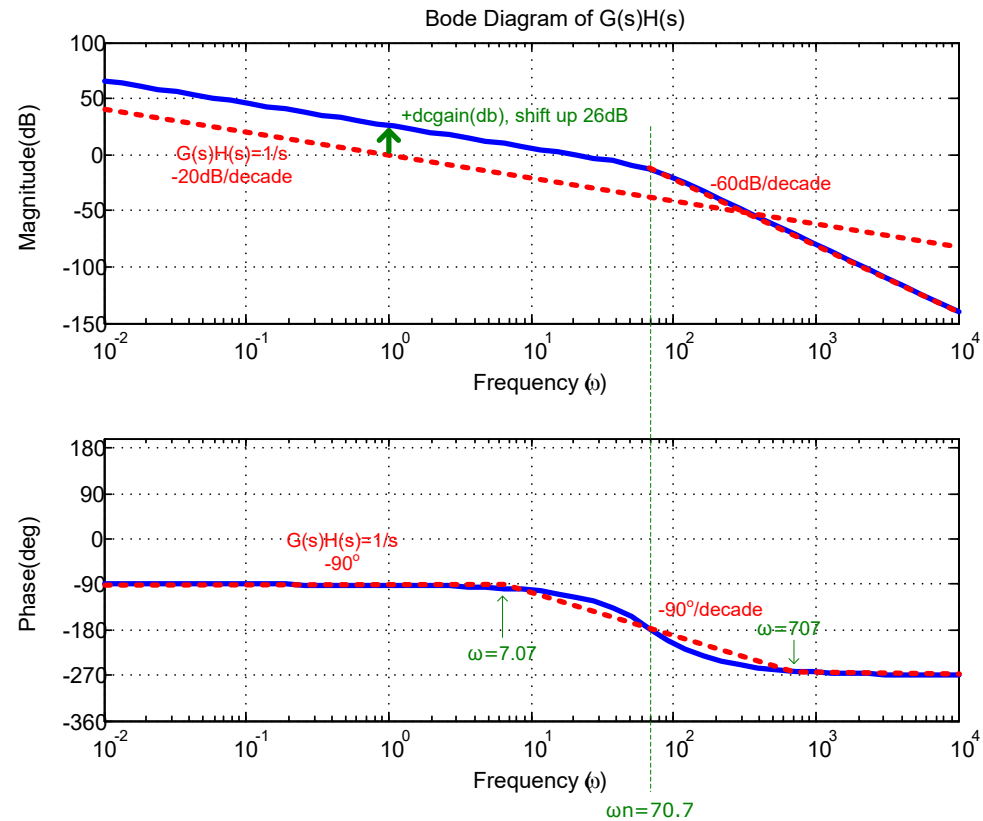
Re - write the  $G(s)H(s)$  into standard form

$$G(s)H(s) = \frac{100000}{(\sqrt{5000})^2} \frac{1}{s} \frac{(\sqrt{5000})^2}{s^2 + 2 \cdot \frac{50}{\sqrt{5000}} \sqrt{5000}s + (\sqrt{5000})^2}$$

$$G(s)H(s) = 20 \cdot \frac{1}{s} \frac{(\sqrt{5000})^2}{s^2 + 2 \times 0.707 \times \sqrt{5000}s + (\sqrt{5000})^2}$$

where  $\omega_n = \sqrt{5000} = 70.7$  and  $\zeta = 0.707$

dcgain = 20  
 dcgain(dB) = 20\*log(dcgain)=26dB



# Frequency Response Method - Nyquist Plot

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10-22-2012

<http://www.kskelvin.net>

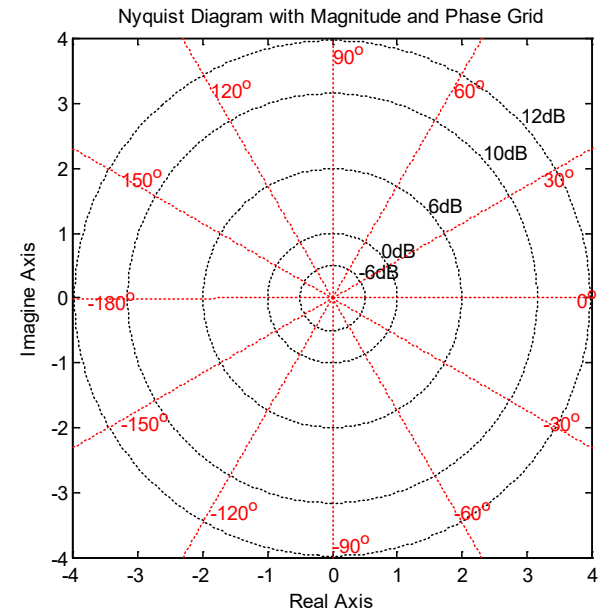
# Nyquist Plot

## □ Introduction

- Bode plot and Nyquist plot are commonly used in the frequency-response representation of LTI (Linear Time Invariant) feedback control systems.

- Bode plot is rectangular plot
- Nyquist plot is polar plot
  - Includes the loci for both  $\omega > 0$  and  $\omega < 0$ .

Black Line: Magnitude of  $G(s)H(s)$   
Red Line: Phase of  $G(s)H(s)$



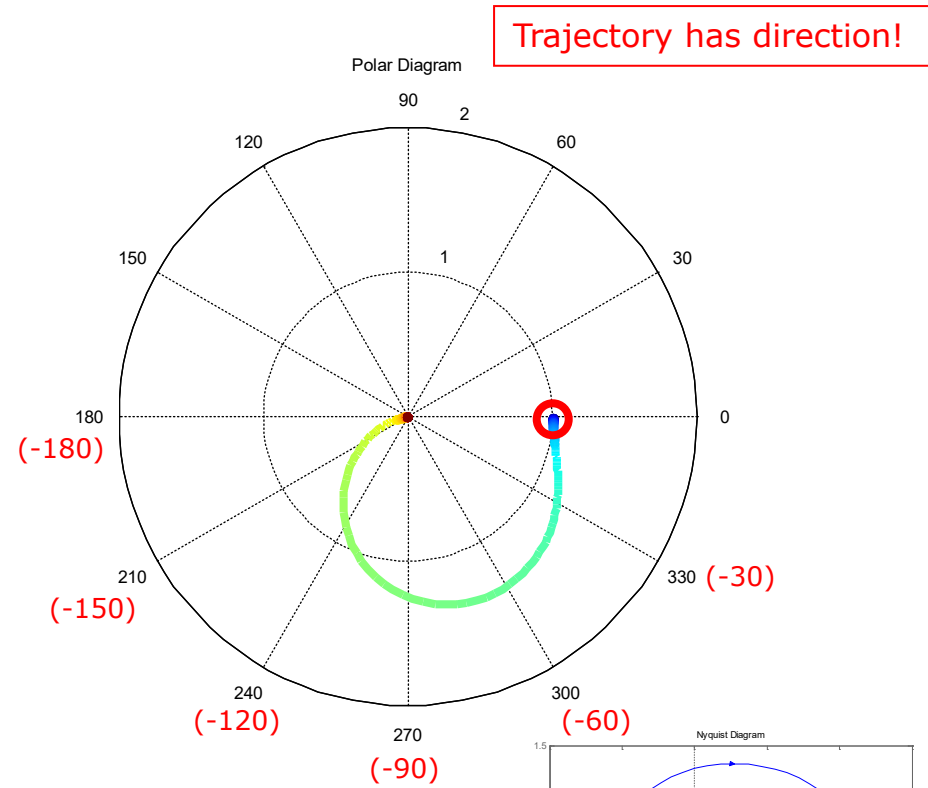
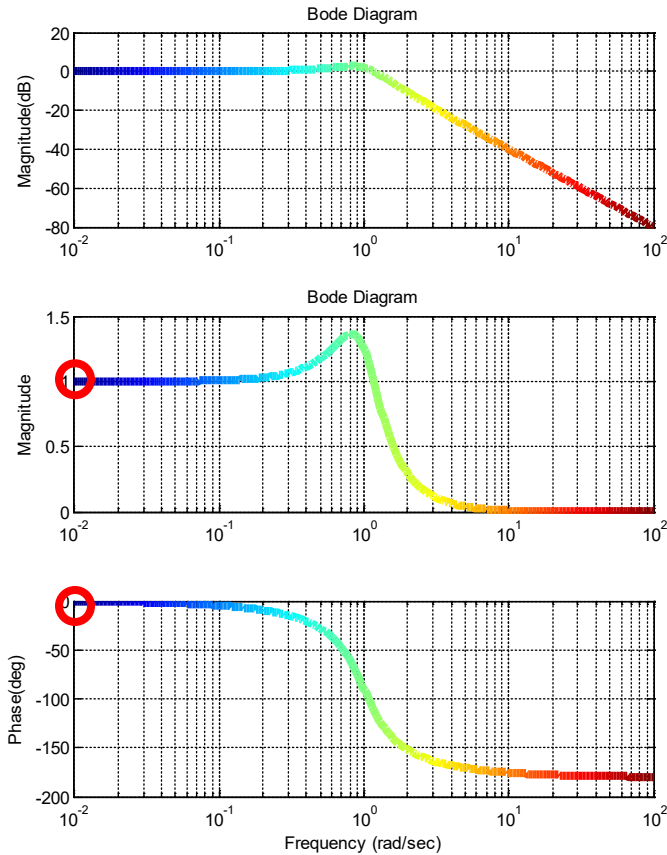
$$Mag_{dB} = 20 \log_{10}(Mag)$$

or

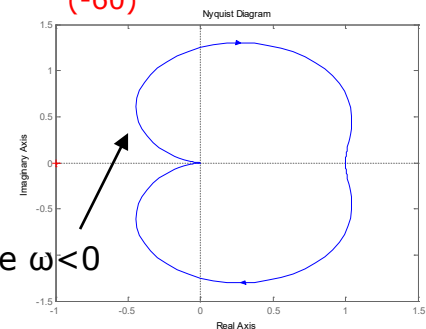
$$Mag = 10^{\frac{Mag_{dB}}{20}}$$



# Bode and Polar plots

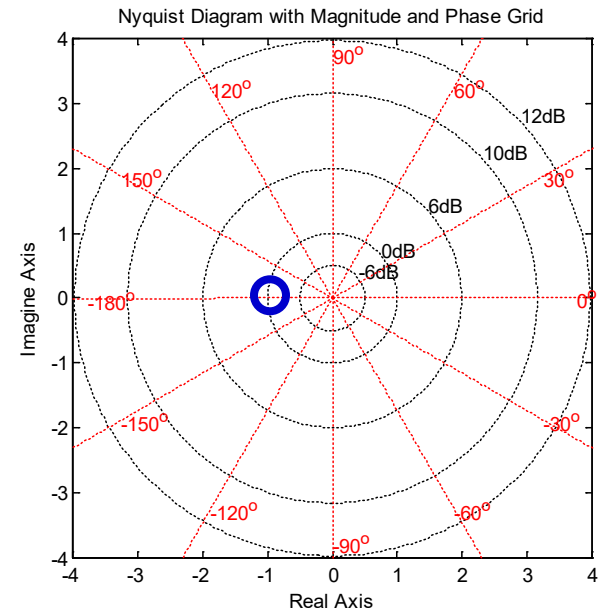


Nyquist plot include  $\omega < 0$



# Special Point in Nyquist Plot

- Point:  $-1+j0$ 
  - Magnitude = 1 = 0dB
  - Phase =  $-180^\circ$
  - In Bode plot, for a stable system
    - Condition 1
      - If  $G(s)H(s)$  doesn't have right half plane poles
    - Condition 2
      - $|G(s)H(s)| < 0\text{dB}$  when Phase =  $-180^\circ$ . Therefore, this point is critical in Nyquist plot.



# Stability Analysis of Nyquist plot

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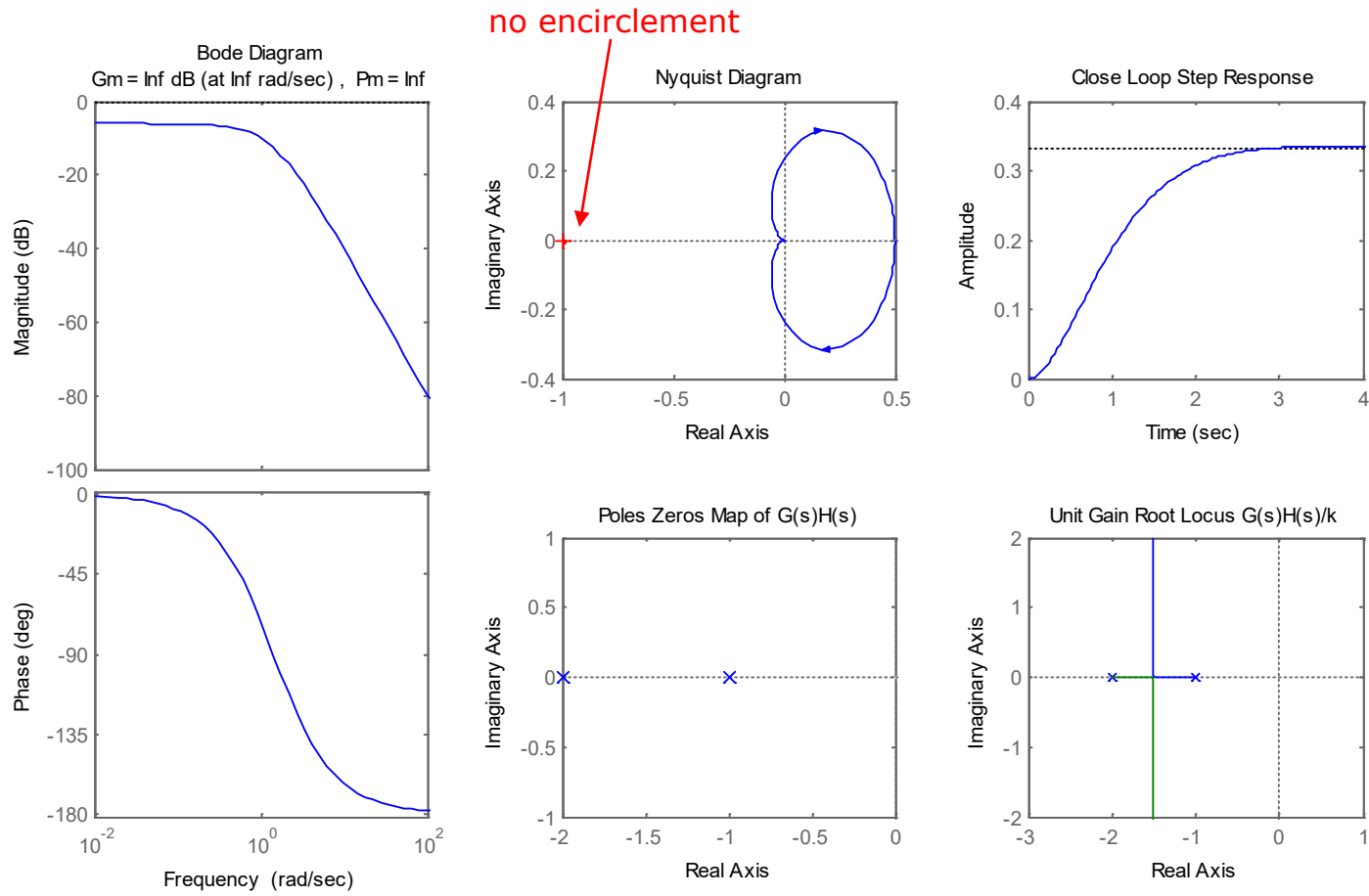
## □ Reference

- Page 454, "Modern Control Engineering (5th Edition)", Katsuhiko Ogata.

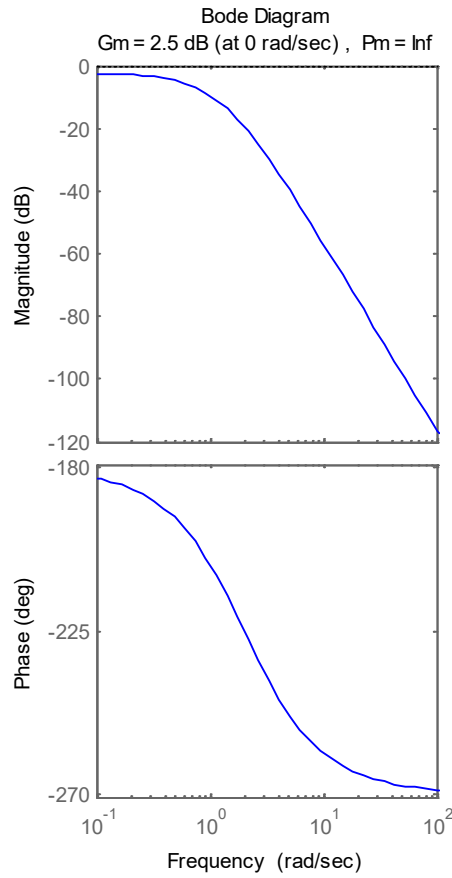
## □ Rules

1. There is **no encirclement of the  $-1+j0$  point**. This implies that the system is stable if there are no poles of  $G(s)H(s)$  in the right-half  $s$  plane; otherwise, the system is unstable.
2. There are **one or more counterclockwise encirclements of the  $-1+j0$  point**. In this case the system is stable if the number of counterclockwise encirclements is the same as the number of poles of  $G(s)H(s)$  in the right-half  $s$  plane; otherwise, the system is unstable.
3. There are **one or more clockwise encirclements of the  $-1+j0$  point**. In this case the system is unstable.

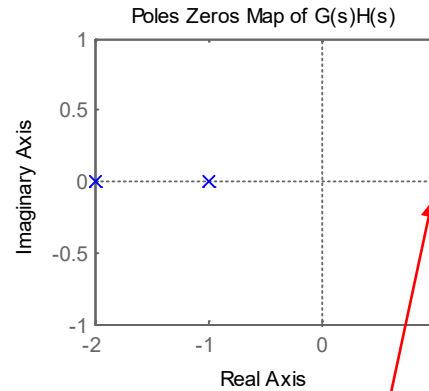
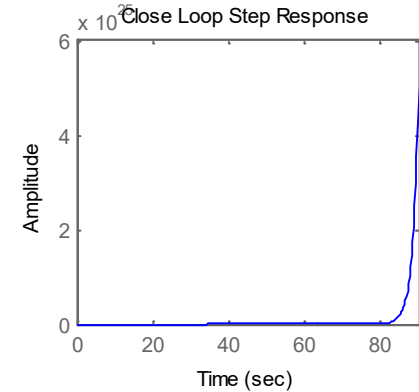
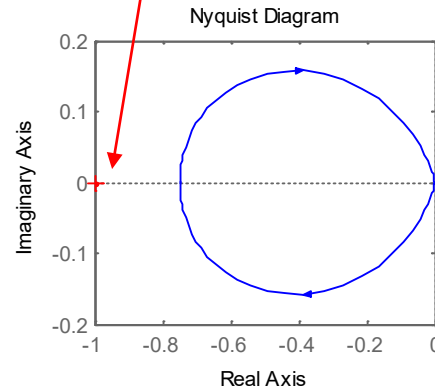
# Rule #1, Stable System (no encirclement of $-1+j0$ )



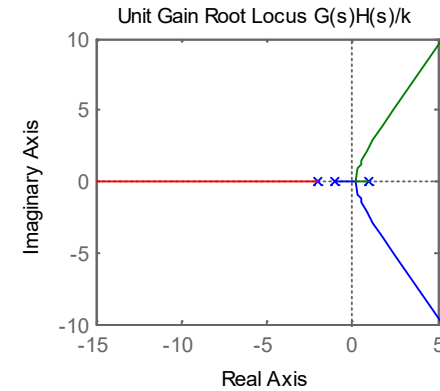
# Rule #1, Unstable System (no encirclement of $-1+j0$ )



no encirclement

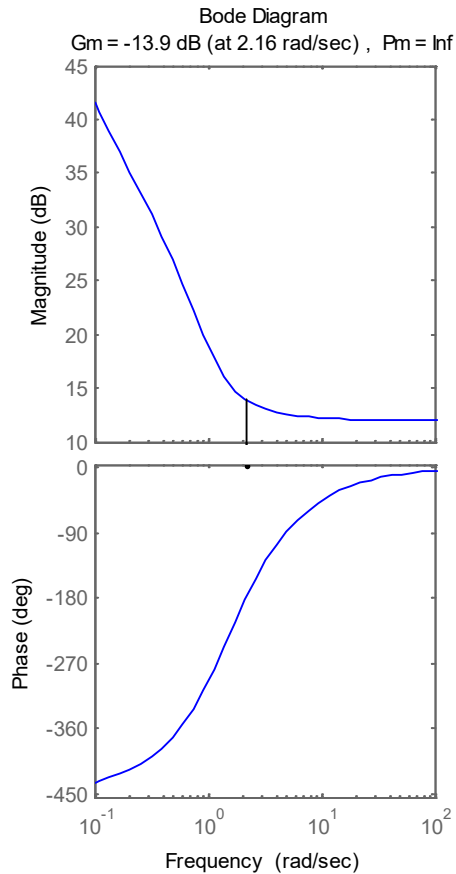


RHP poles

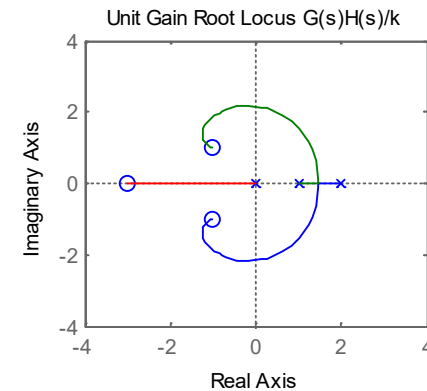
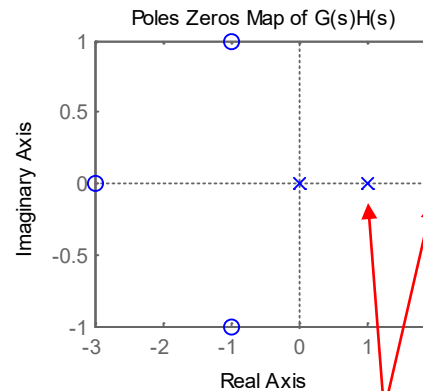
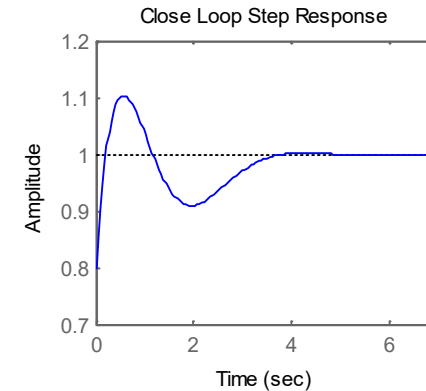
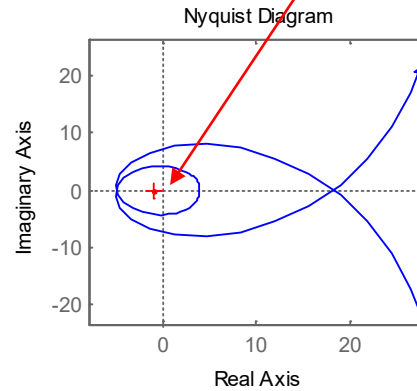


# Rule #2, Stable System

(counterclockwise encirclement of  $-1+j0$ )



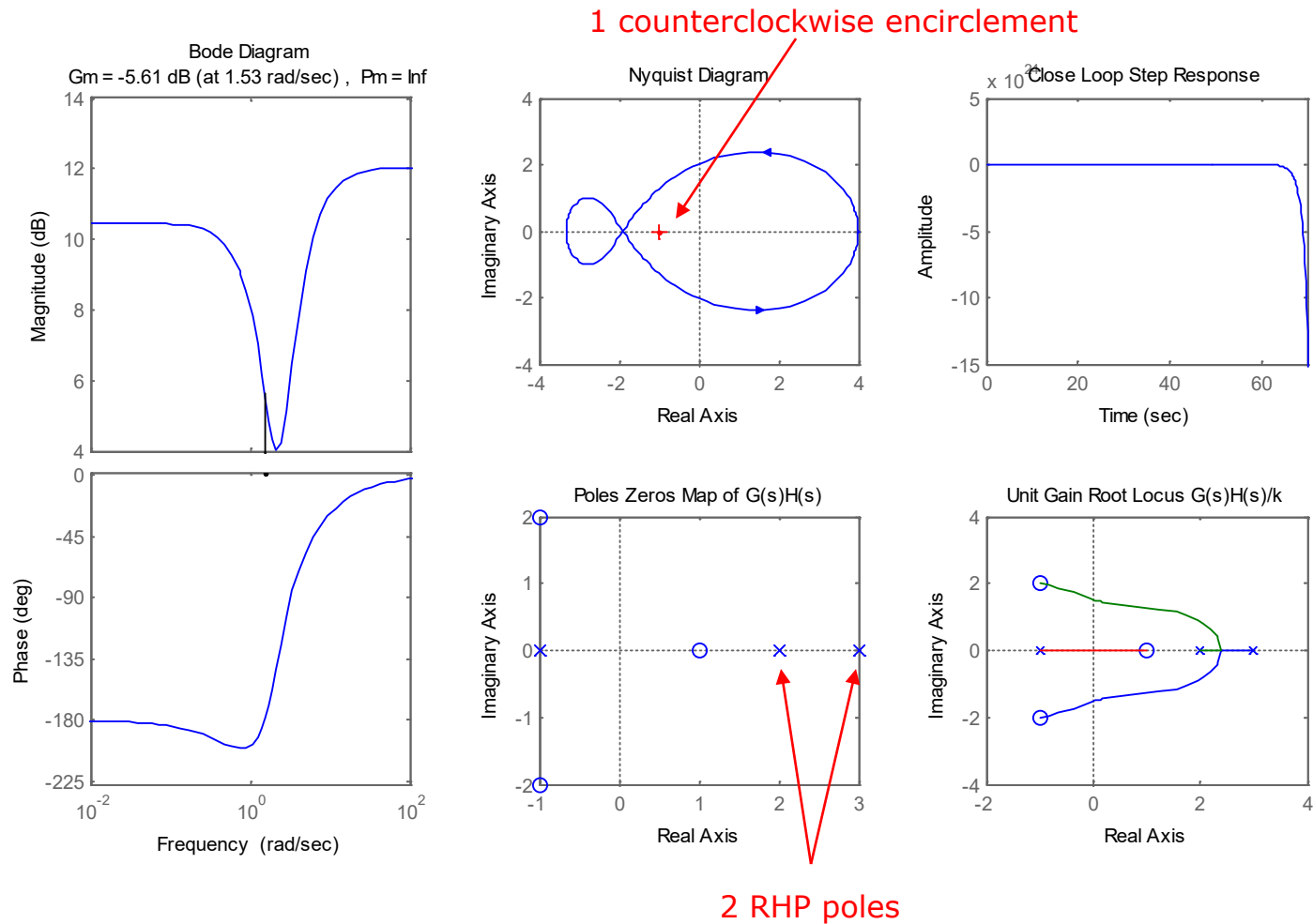
2 counterclockwise encirclement



2 RHP poles

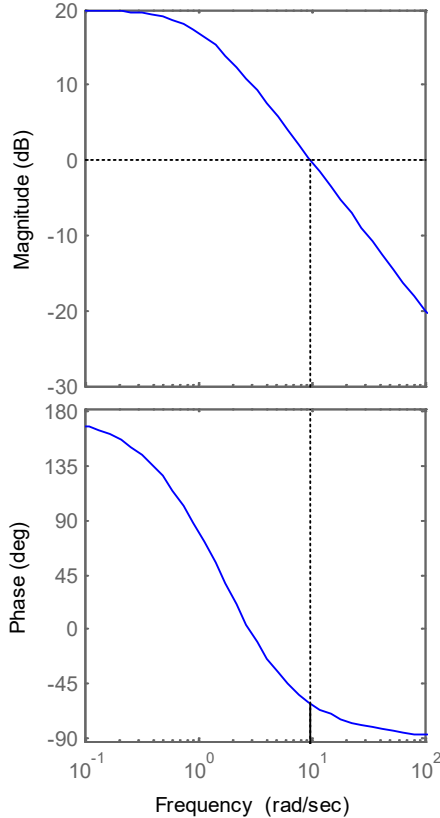
# Rule #2, Unstable System

## (counterclockwise encirclement of $-1+j0$ )



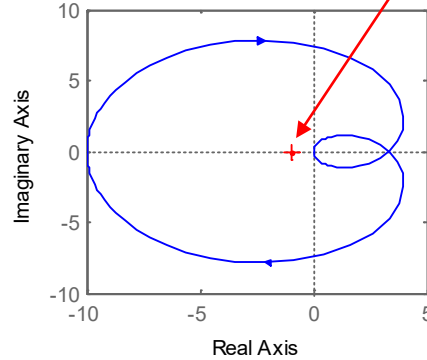
# Rule #3, Unstable System (clockwise encirclement of $-1+j0$ )

Bode Diagram  
Gm = -20 dB (at 0 rad/sec), Pm = 118 deg (at 9.95 rad/sec)

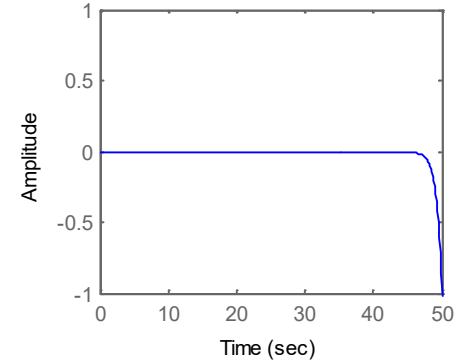


1 clockwise encirclement

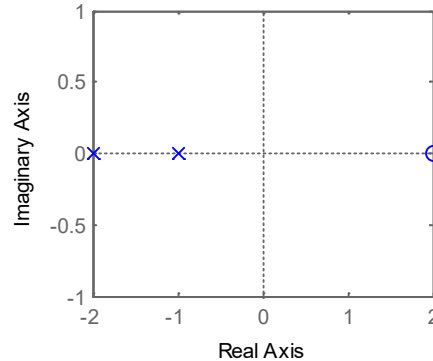
Nyquist Diagram



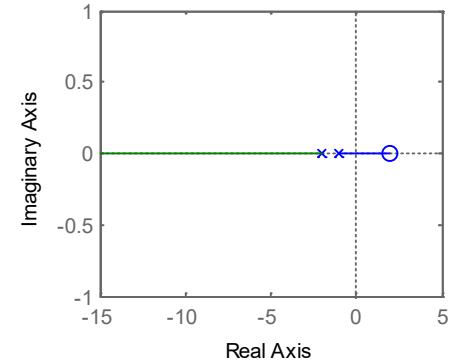
Close Loop Step Response




Poles Zeros Map of  $G(s)H(s)$



Unit Gain Root Locus  $G(s)H(s)/k$

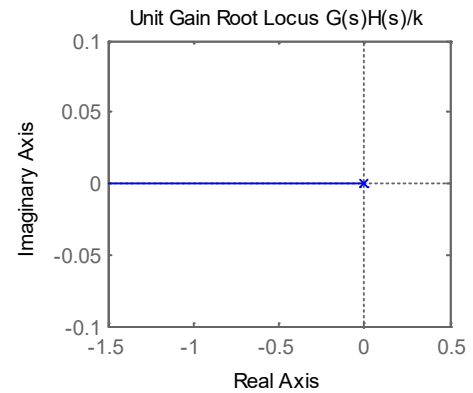
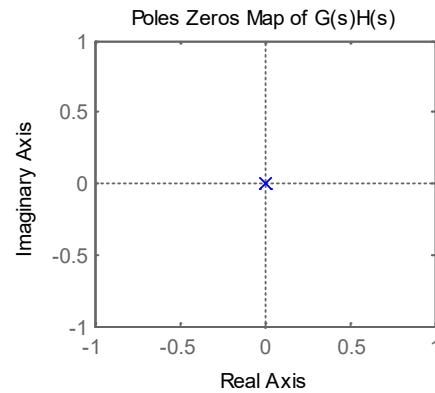
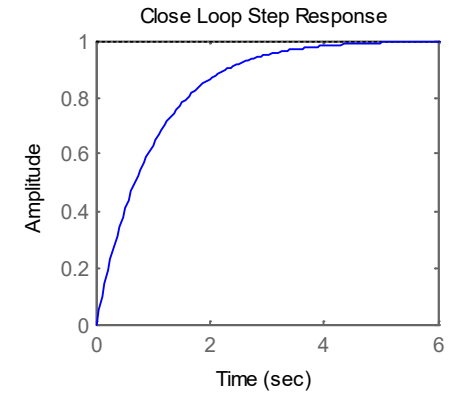
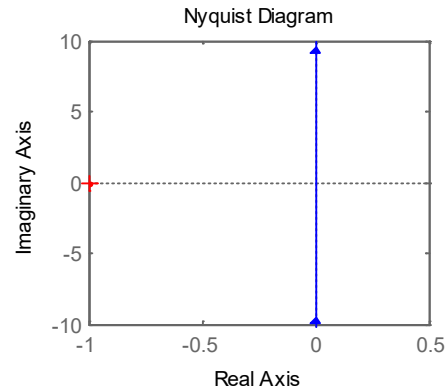
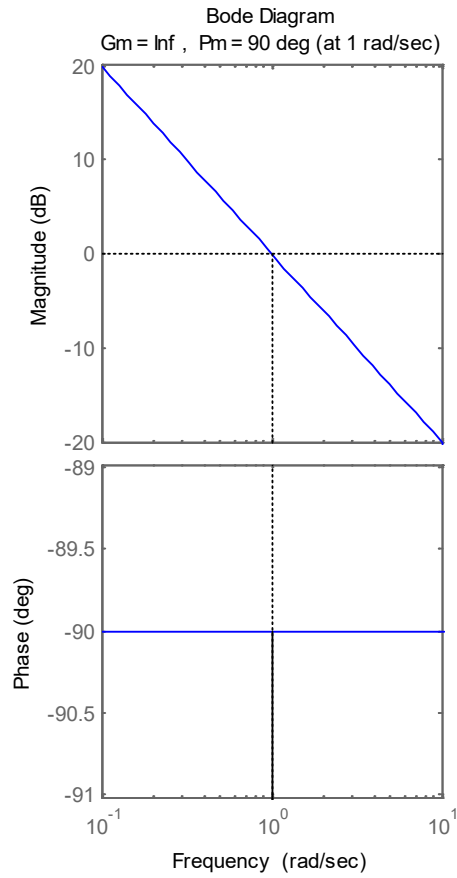




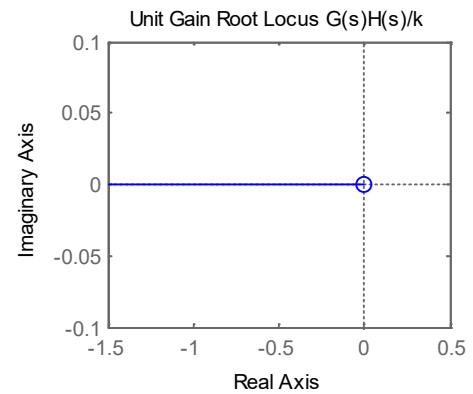
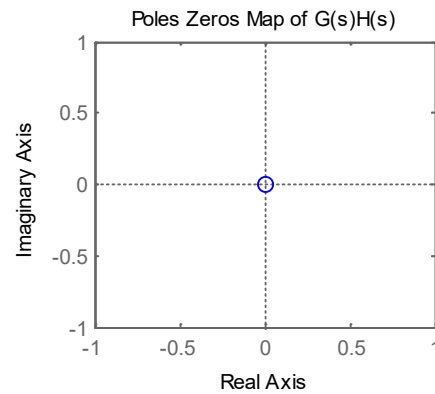
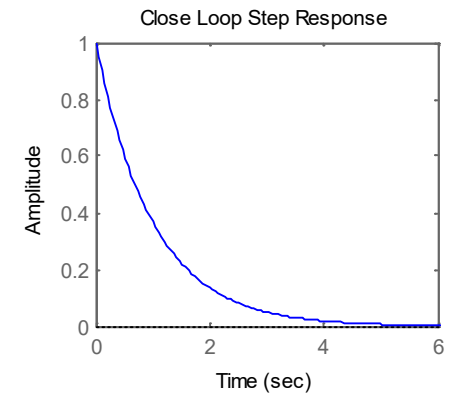
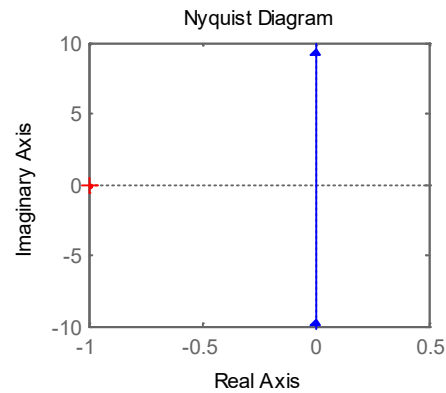
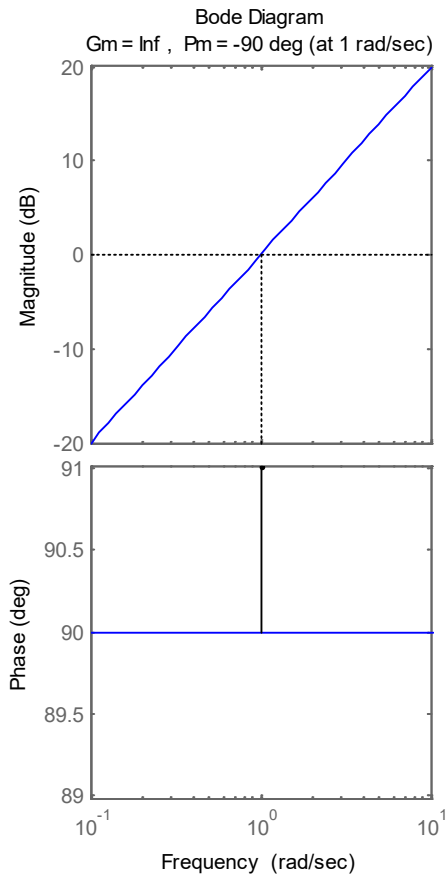


Appendix  
Standard  $G(s)H(s)$  Bode and  
Nyquist Plots

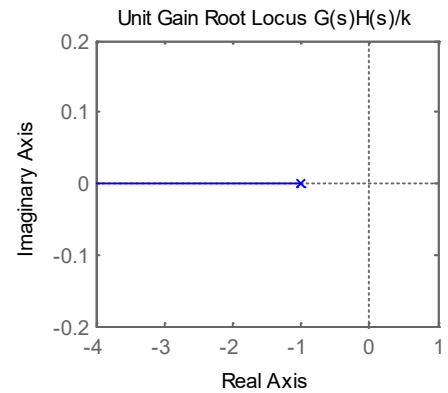
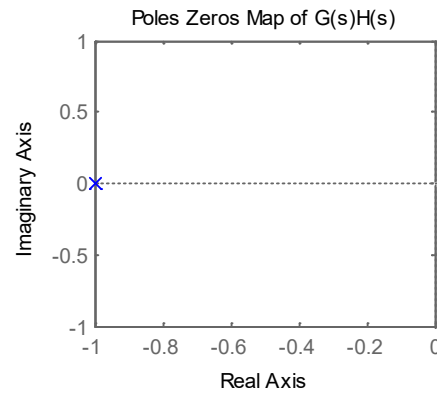
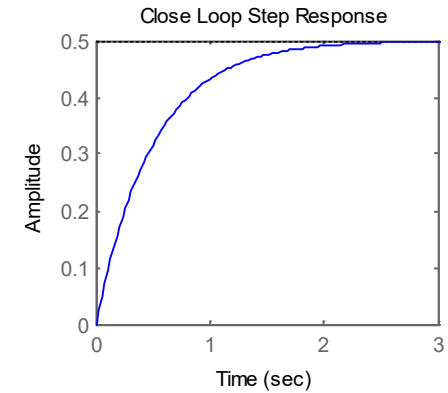
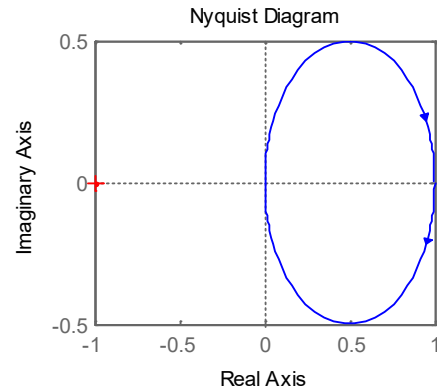
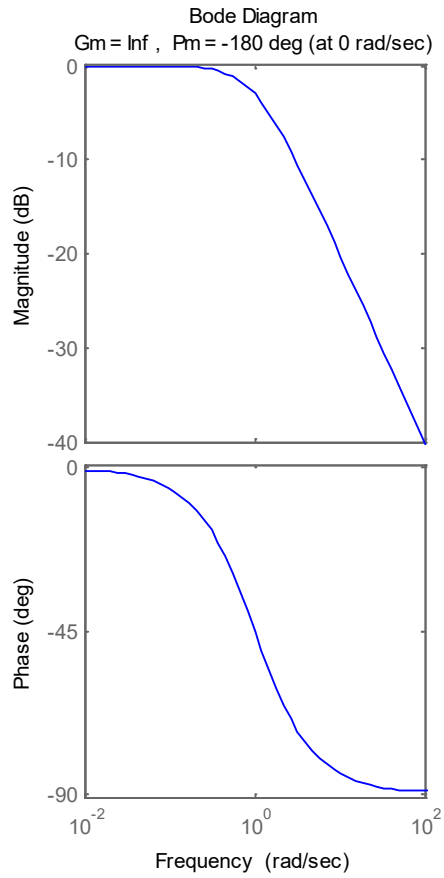
$$G(s)H(s) = \frac{1}{s}$$



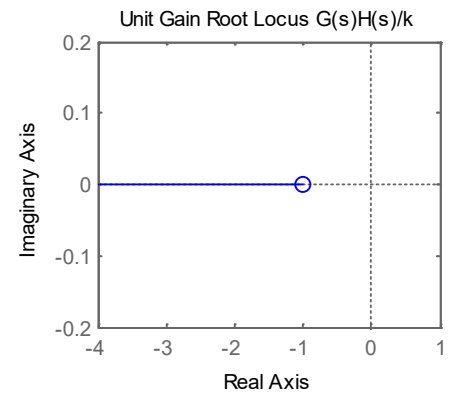
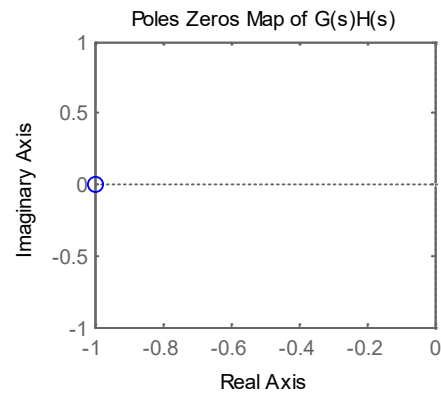
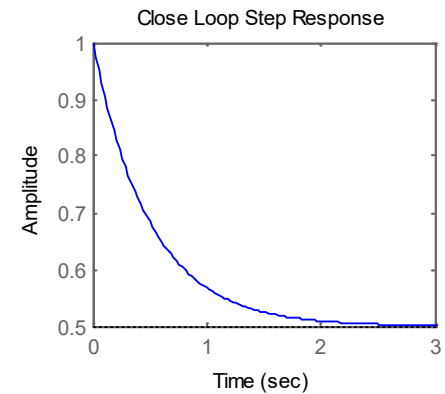
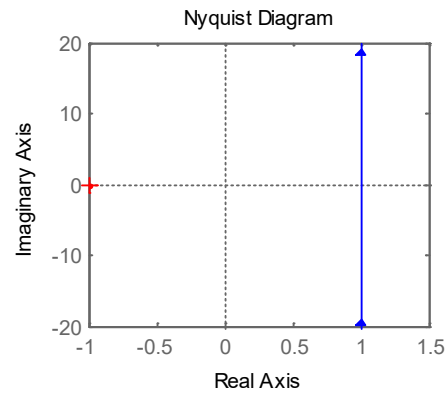
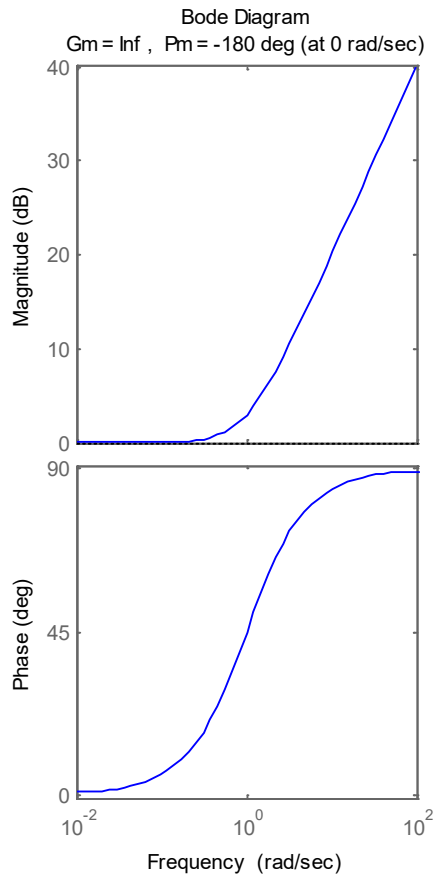
$$G(s)H(s) = s$$



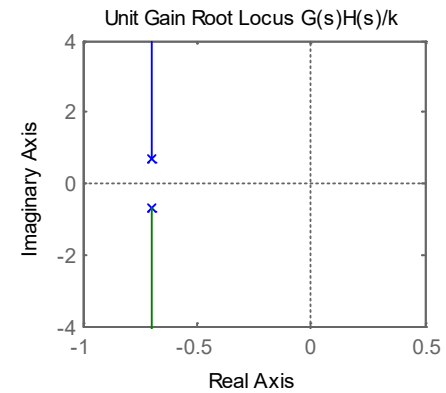
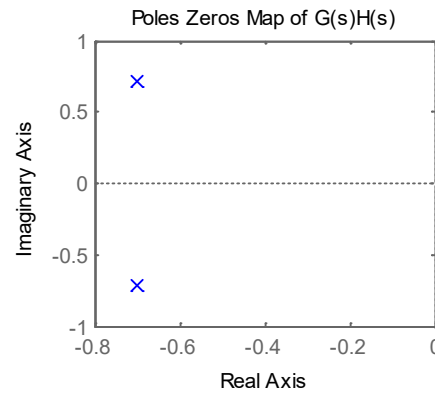
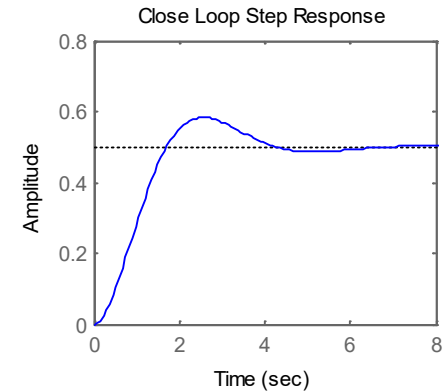
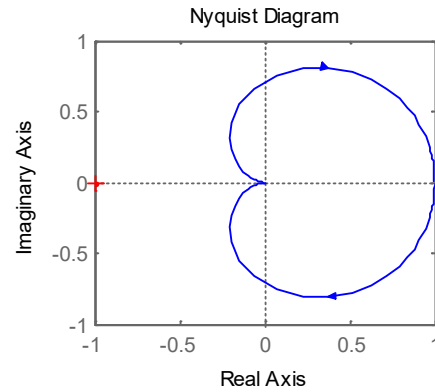
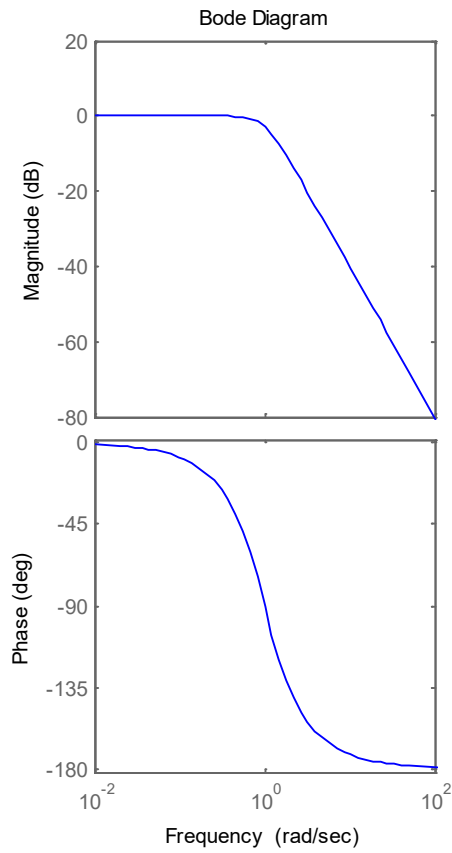
$$G(s)H(s) = \frac{1}{s+1}$$



$$G(s)H(s) = s + 1$$



$$G(s)H(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \text{ where } \omega_n = 1, \zeta = 0.7$$



# Gain and Phase Margins



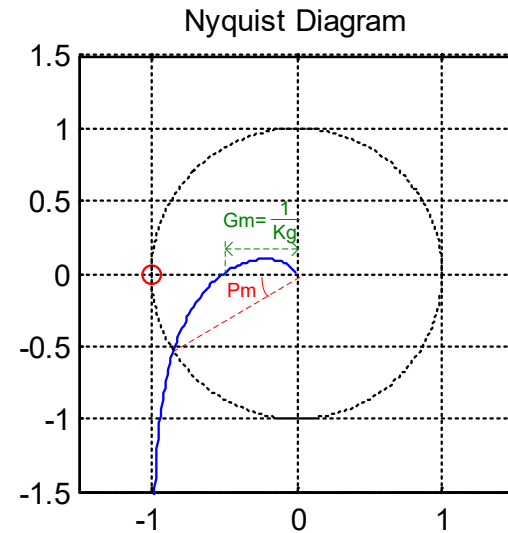
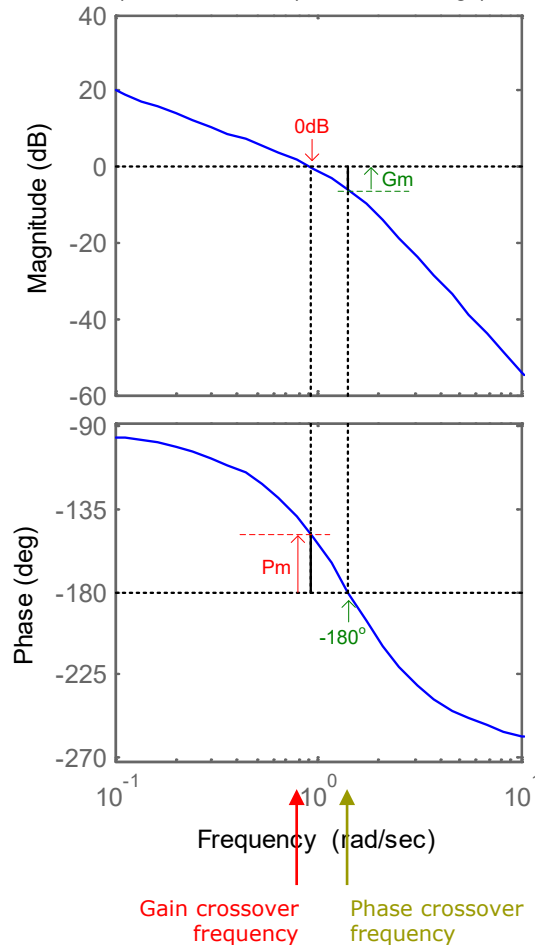
Kelvin Leung, Ph.D.

10-22-2012

<http://www.kskelvin.net>

# Gain and Phase Margins Definition in Bode and Nyquist plots (stable)

Bode Diagram  
 $G_m = 6.02 \text{ dB}$  (at  $1.41 \text{ rad/sec}$ ),  $P_m = 32 \text{ deg}$  (at  $0.921 \text{ rad/sec}$ )



Numerical Example for Nyquist plot

$$K_g = 0.5$$

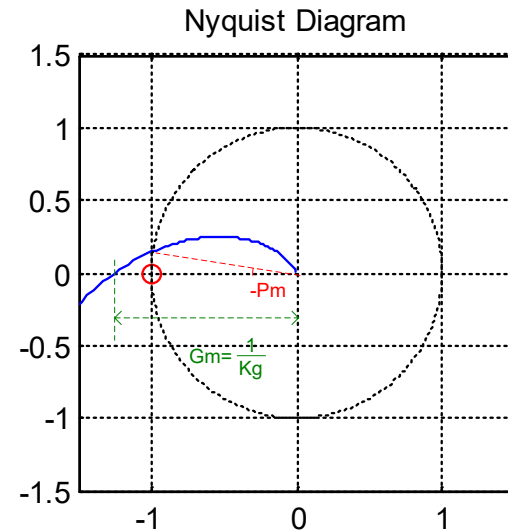
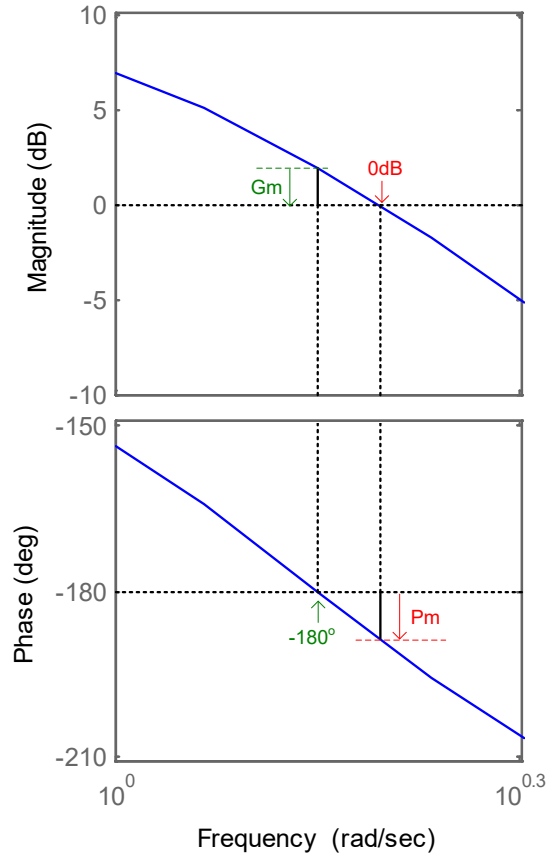
$$G_m = \frac{1}{K_g} = 2 = 20 \log(2) = 6.02 \text{ dB}$$

$$P_m = \tan^{-1}\left(\frac{0.5305}{0.8476}\right) = 32^\circ$$



# Gain and Phase Margins Definition in Bode and Nyquist plots (unstable)

Bode Diagram  
 $G_m = -1.94$  dB (at 1.41 rad/sec) ,  $P_m = -8.54$  deg (at 1.57 rad/sec)



Numerical Example for Nyquist plot

$$K_g = 1.25$$

$$G_m = \frac{1}{K_g} = 0.8 = 20 \log(0.8) = -1.94 \text{ dB}$$

$$P_m = -\tan^{-1}\left(\frac{0.1484}{0.9889}\right) = -8.54^\circ$$

# Relationship between open-loop and close-loop response

---

## □ Natural frequency ( $\omega_n$ )

- $\omega_n$  in closed-loop system is somewhere between the gain crossover frequency and phase crossover frequency in open-loop system.
  - page. 473-474 of "Modern Control Engineering", Ogata, 5th Edition
- A very rough estimate is that the bandwidth (freq @ -3dB) is approximately equal to the natural frequency.
  - [<http://www.engin.umich.edu/class/ctms/freq/freq.htm>]

## □ Damping ratio ( $\zeta$ )

- Phase margin in open-loop system has linear relationship with  $\zeta$  of closed-loop system
  - Exact Formula

Phase margin ( $\gamma$ ) and Damping Factor ( $\zeta$ )

$$\gamma = \tan^{-1} \frac{2\zeta}{\sqrt{\sqrt{1+4\zeta^4} - 2\zeta^2}}$$

- Approximation: for  $\zeta < 0.6$ ,  $\zeta = 0.01 \text{ Pm}$

# Design with Bode Plot

---

## □ Design criteria

- Bode plot can only be used to design close-loop feedback from stable open-loop system (i.e.  $G(s)H(s)$  doesn't has RHP poles), otherwise, Nyquist or Root-Locus need to be used.

- Reason: Refer to stability rule #1 in Nyquist plot powerpoint.

## ■ System performance

### □ DC Gain

- Determine the steady state error
  - Increase of DC Gain, Decrease of Steady State Error

### □ Phase margin

- Determine the damping ratio and overshoot.
  - Phase margin is normally selected to between  $30^\circ$ - $60^\circ$ .

### □ Gain margin

- Determine the robustness of system. Normally  $> 6\text{dB}$ .
  - To guarantee stability even if the open-loop gain and time constants of the components vary to a certain extend.

### □ Gain/Phase crossover frequency

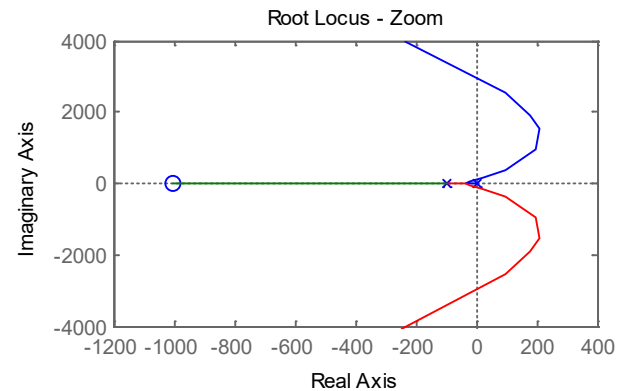
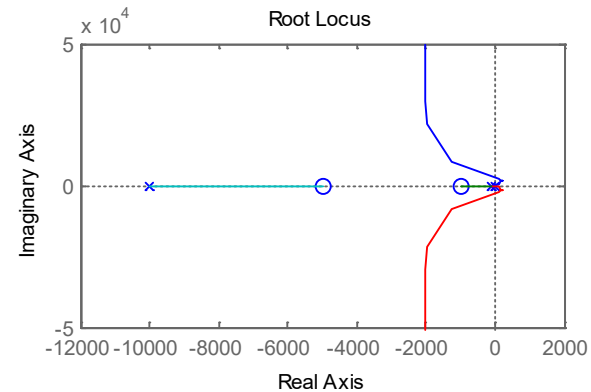
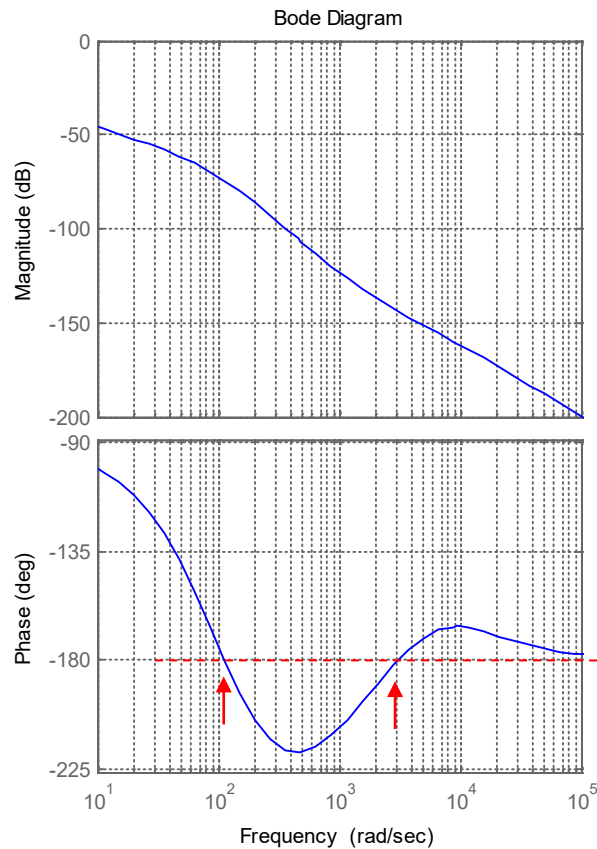
- Determine the transient response speed.
  - Increase the crossover frequency, Increase transient speed.



# Stability of multiple phase crossover frequencies system

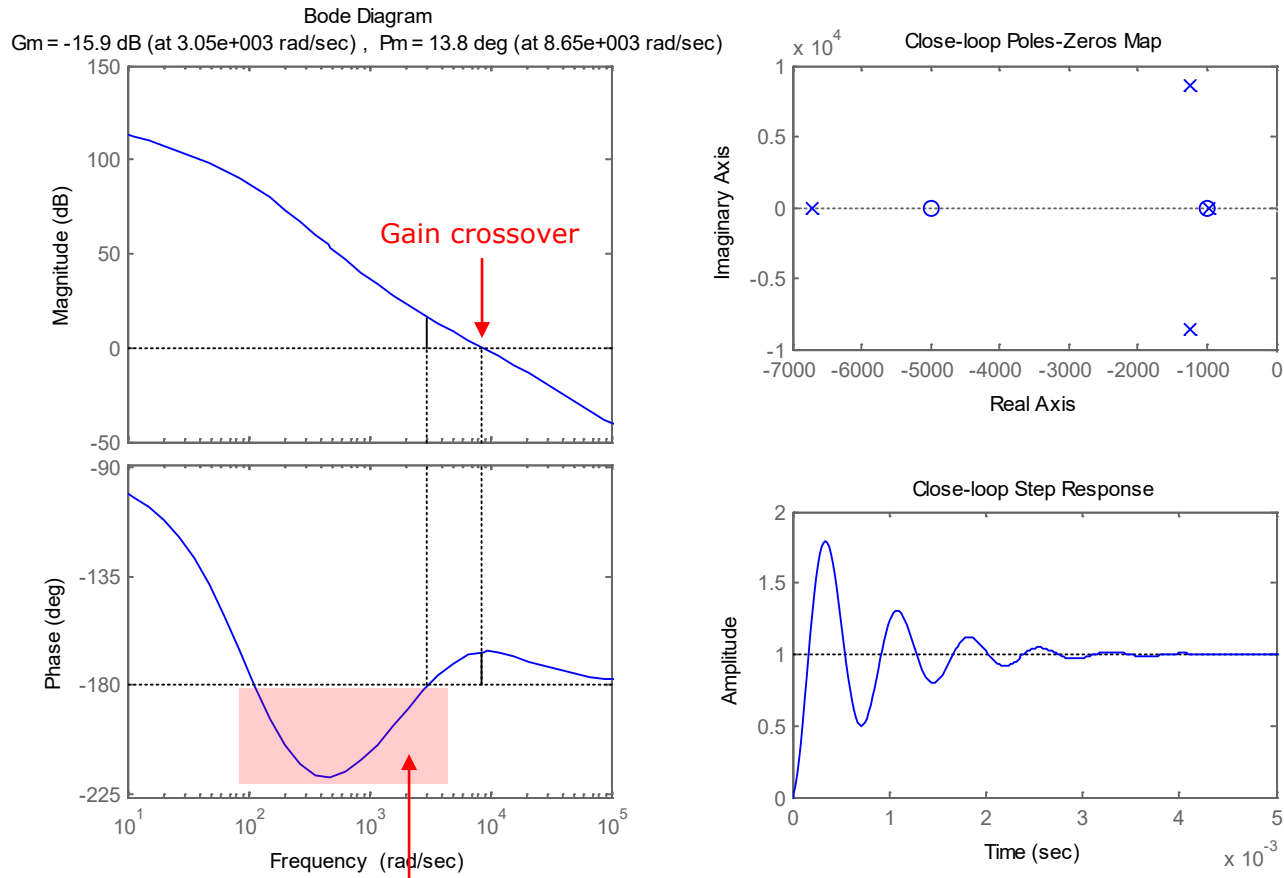
# Stability of multiple phase crossover

Bode plot shows a system which has multiple phase crossover at  $180^\circ$



Root locus shows that there are 2 region of gain K which can give stable system  
Therefore, root locus actually indicate "second" phase crossover can be used to generate a stable system

# Stability of multiple phase crossover

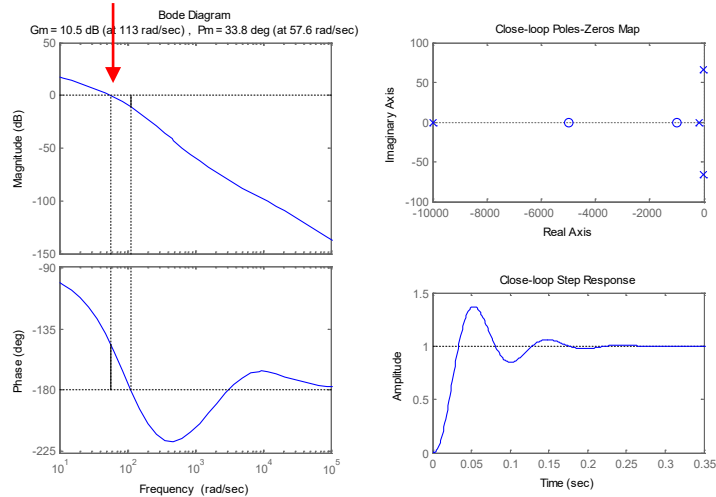


System is stable as long as phase margin is +ve, even the phase drops below  $-180^\circ$  before that.

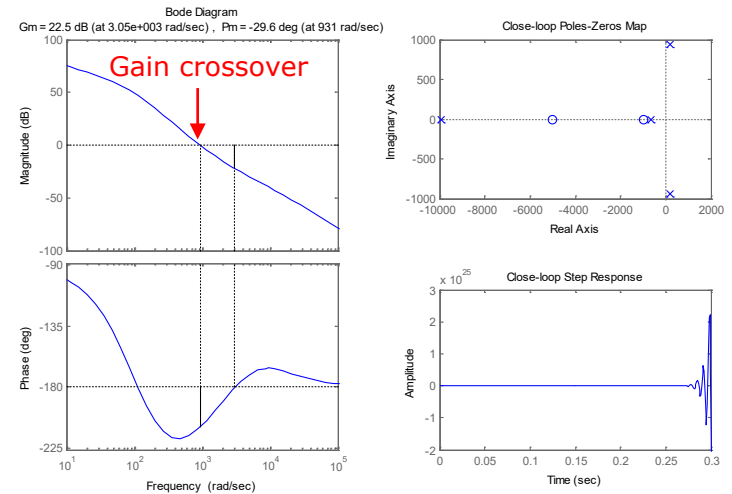
# Stability of multiple phase crossover

## Stable Case: Low Gain

Gain crossover



## Unstable Case: Middle Gain



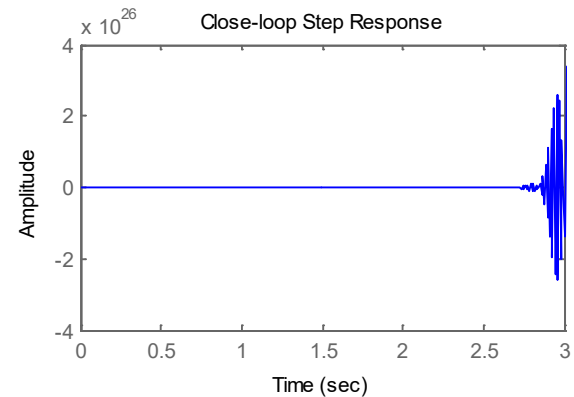
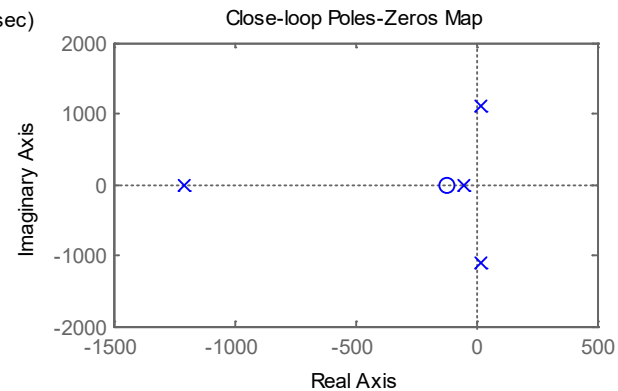
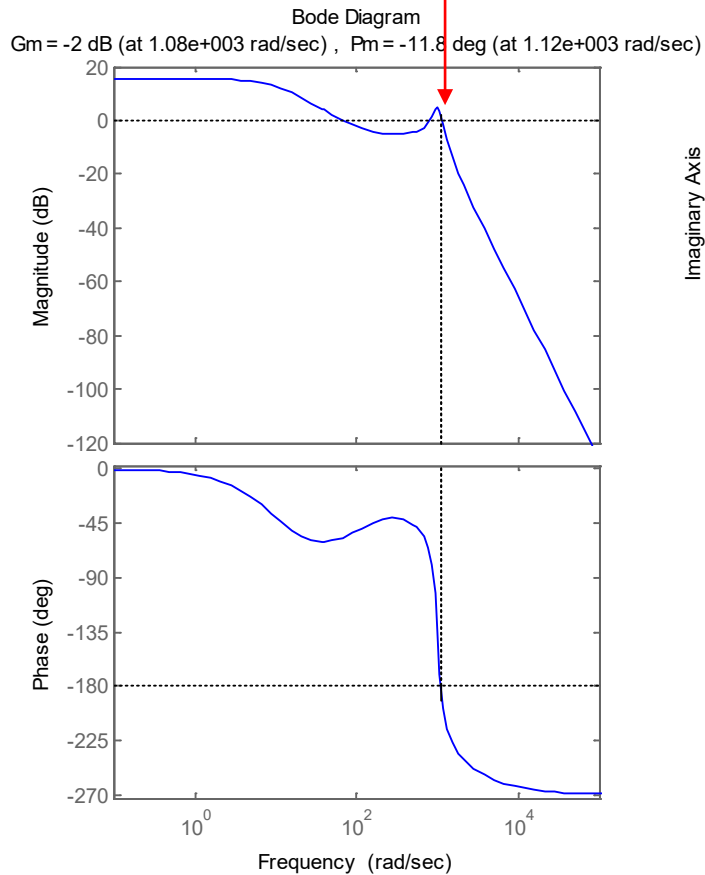


# Stability of multiple gain crossover frequencies system



# Stability of multiple gain crossover

Phase margin is measured at the highest gain crossover frequency



# Design with Lead or Lag Compensator

---

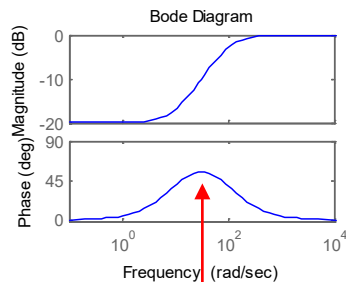
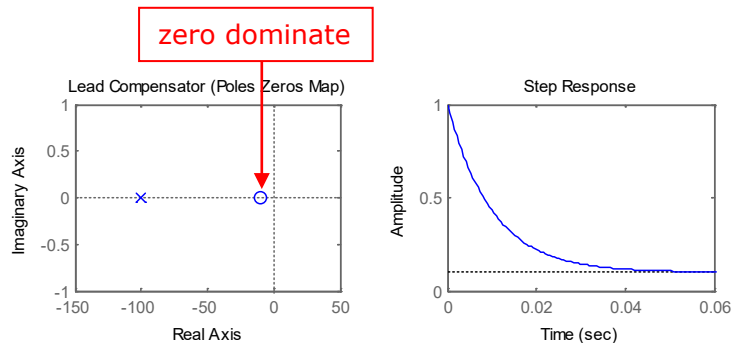
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10-19-2012

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# Lead and Lag Compensator Definition

## Lead Compensator



$$G_C(s) = \frac{s + z}{s + p}$$

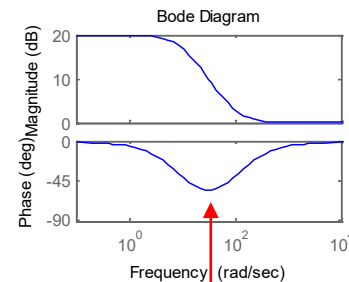
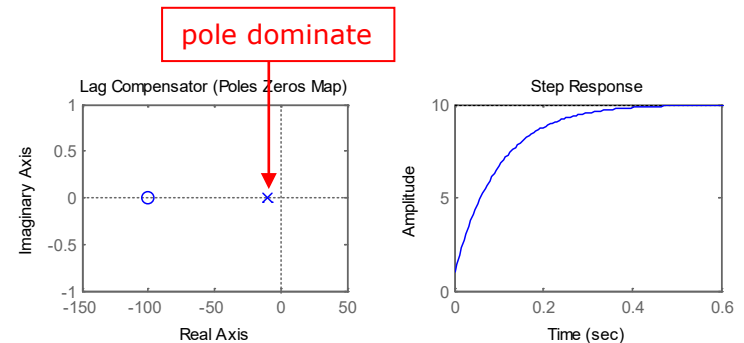
where  $p > z$

Add +ve phase

Application:

- Speed up system response

## Lag Compensator



$$G_C(s) = \frac{s + z}{s + p}$$

where  $z > p$

Subtract -ve phase

Application:

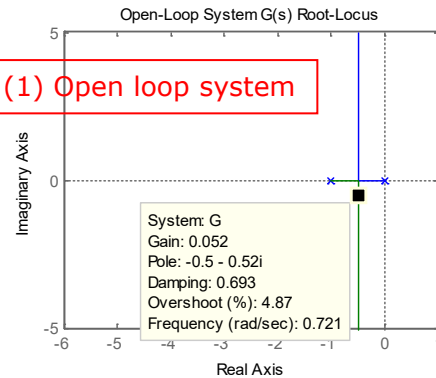
- Improve steady state error
- Avoid changing system dynamic



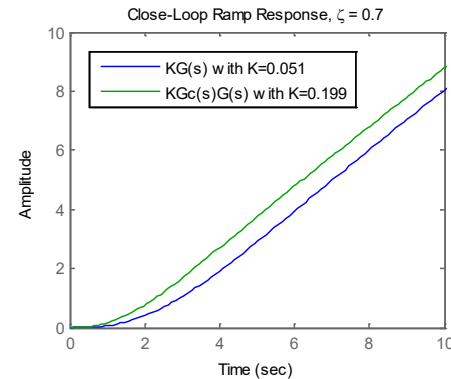
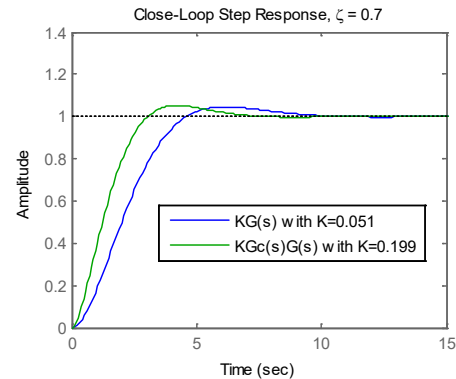
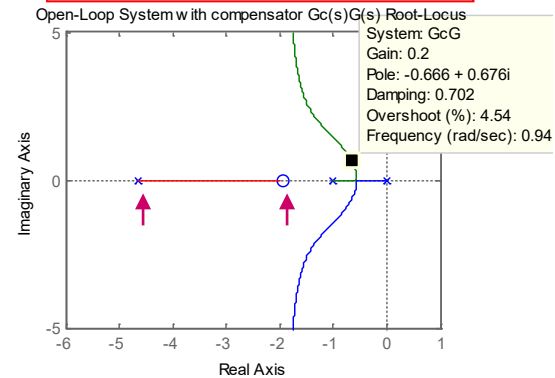
Design with lead compensator

# Example of system compensation with lead compensator – root locus

(1) Open loop system



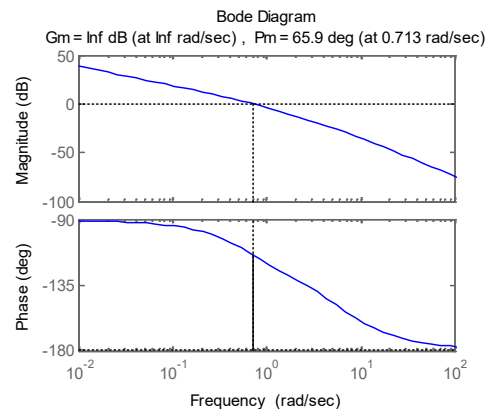
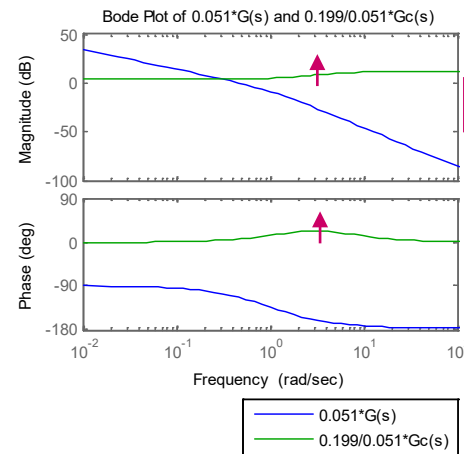
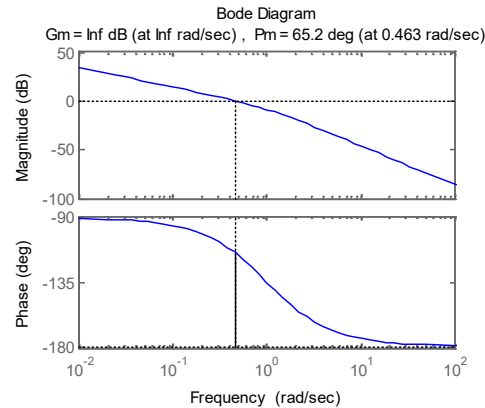
(2) Add a lead compensator to shift the root locus to left



(3) Select  $K$  with  $\zeta=0.7$  for comparison

1. Step and ramp response improved
2. System speed increased

# Example of system compensation with lead compensator – bode plot



## Consideration

1. To improve the speed, we need to boost the gain for higher crossover frequency
2. However, if we only boost up the gain, phase margin reduce.
3. Lead compensator can boost up the gain and phase.
4. Therefore, crossover frequency increased without changing phase margin

## Remark:

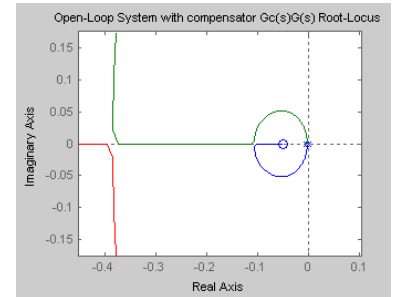
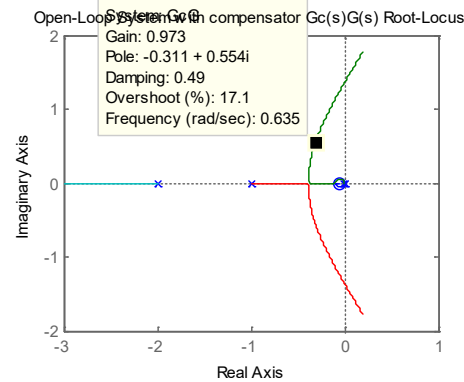
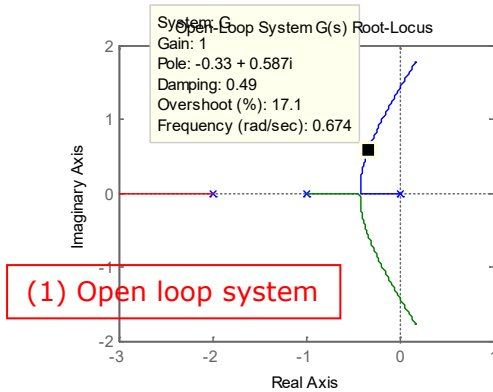
- crossover frequency related to  $\omega_n$ .  $\omega_n$  is somewhere between gain crossover freq and phase crossover freq.
- phase margin related to  $\zeta$



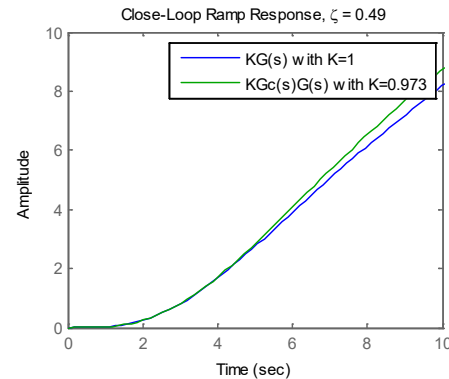
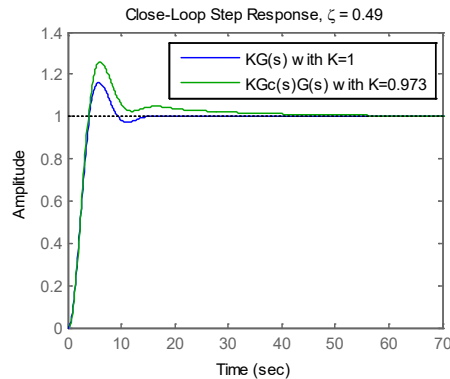
Design with lag compensator

# Example of system compensation with lag compensator – root locus

(2) Add a lag compensator



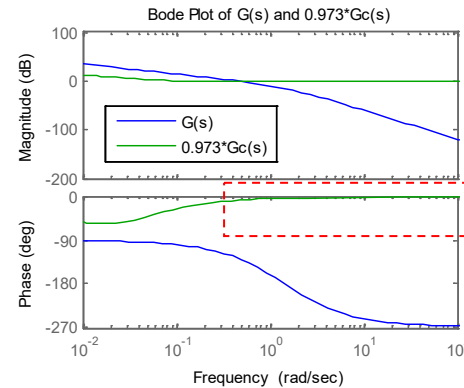
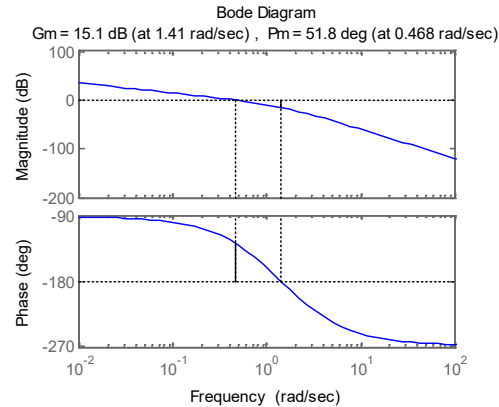
Zoom near Origin



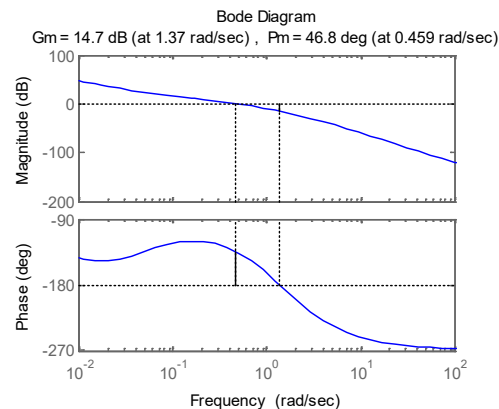
- (3) Select  $K$  with  $\zeta=0.49$  for comparison
1. Avoid changing system dynamic
  2. Reduce steady state error at ramp response



# Example of system compensation with lag compensator – bode plot



Lag compensator  
doesn't affect phase in  
high frequency region



## Consideration

1. To improve steady state response (ramp input) but avoid changing system dynamic, we need to boost the gain at low frequency without changing crossover frequency and phase margin
2. By properly select the Lag compensator, it can increase the gain at low frequency without affecting the phase near crossover frequency.
3. Therefore, low frequency gain increased without changing crossover characteristic.

# Analog to Digital Implementation



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24-10-2012

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# Analog to Digital Implementation

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## □ Purpose

- This ppt is intended to show the procedure of transforming an analog system into digital implementation with the help of matlab.
- Theory of digital control is not the target of this ppt.
- By following this ppt, you can convert your analog compensator into a digital formula and implement it in a digital processor.

# Concept of Sampling

## □ Concept of Sampling

- In digital controller, ADC (analog-to-digital converter) and DAC (digital-to-analog converter) are used. ADC and DAC are not continuous device but discrete time sampling input or output.
- Sampling frequency is determined by designer. This is an important parameters to interface the analog and digital system.

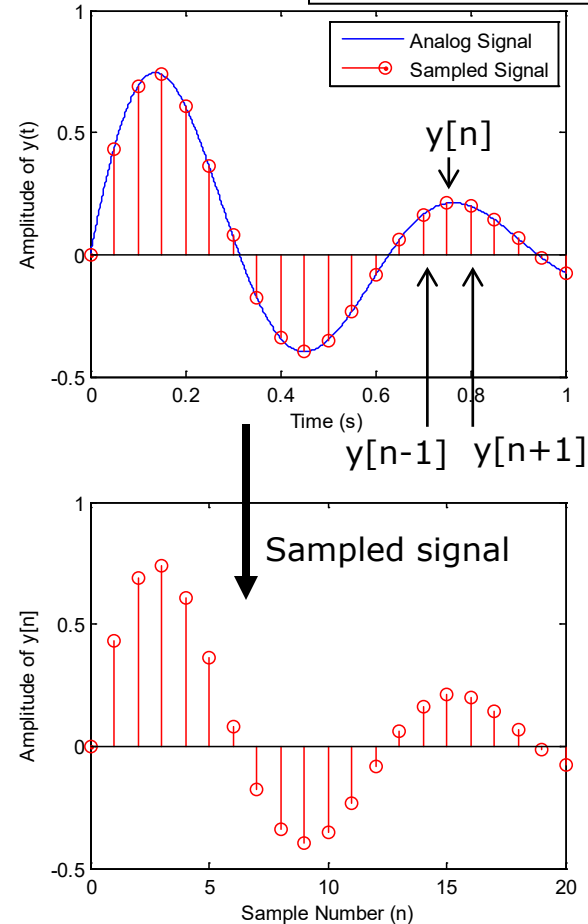
## □ Concept of Z-transform

- S-domain transfer function can be converts into z-domain through z-transform.
- In z-transform, we need to remember time-shifting property

$$x[n - k] = z^{-k} X(z)$$

Sampling frequency

$$f_{\text{sampling}} = \frac{1}{T_{\text{sampling}}} = \frac{1}{0.05} = 20\text{Hz}$$



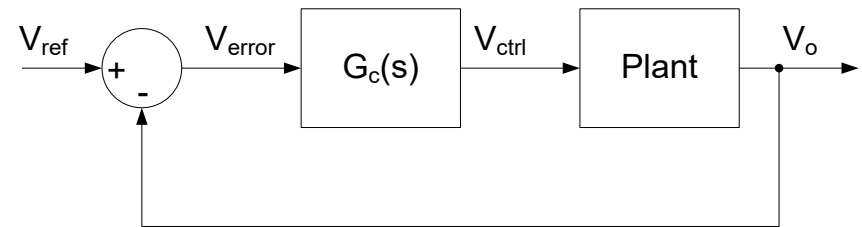
# 2nd-order transfer function example (analog transfer function)

- Assume we design a compensator  $G_c(s)$  and need to implement with a digital controller

$$G_C(s) = \frac{V_{ctrl}(s)}{V_{error}(s)} = \frac{1}{s^2 + 1.4s + 1}$$

- In matlab,

- % define a 2-nd order analog system
- wn=1;
- zeta=0.7;
- num=[wn.^2];
- den=[1 2\*wn\*zeta wn^2];
- G=tf(num,den)



Transfer function:

1

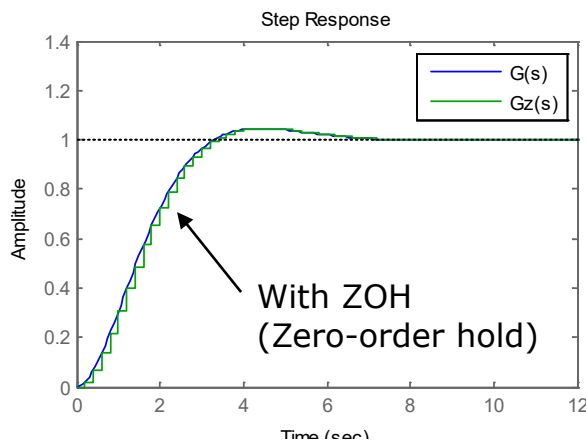
-----  
s^2 + 1.4 s + 1

Define the analog transfer function

# 2nd-order transfer function example (convert to digital transfer function)

## □ In matlab

- % convert the analog system to digital system
- `Gz=c2d(G,0.2)` ← Sampling time
- % form of Gz digital implementation
- `M = idpoly(Gz)`
- % plot the step response of analog and digital system
- `step(G); hold on; step(Gz); hold on;`
- `legend('G(s)', 'Gz(s)');`



```
Transfer function:
  0.0182 z + 0.01657
-----
 z^2 - 1.721 z + 0.7558

Sampling time: 0.2
```

```
Discrete-time IDPOLY model: y(t) = [B(q)/F(q)]u(t) + e(t)
B(q) = 0.0182 q^-1 + 0.01657 q^-2

F(q) = 1 - 1.721 q^-1 + 0.7558 q^-2

This model was not estimated from data.
Sampling interval: 0.2
```

# 2nd-order transfer function example (digital formula from transfer function)

## Method #1

Discrete - time transfer function

$$G_C(z) = \frac{v_{ctrl}(z)}{v_{error}(z)} = \frac{0.0182z + 0.01657}{z^2 - 1.721z + 0.7558}$$

$$G_C(z) = \frac{0.0182z + 0.01657}{z^2 - 1.721z + 0.7558} \frac{z^{-2}}{z^{-2}} = \frac{0.0182z^{-1} + 0.01657z^{-2}}{1 - 1.721z^{-1} + 0.7558z^{-2}}$$

Therefore,

$$v_{ctrl}(z) - 1.721z^{-1}v_{ctrl}(z) + 0.7558z^{-2}v_{ctrl}(z) = 0.0182z^{-1}v_{error}(z) + 0.01657z^{-2}v_{error}(z)$$

$$v_{ctrl}(z) = 0.0182(z^{-1}v_{error}(z)) + 0.01657(z^{-2}v_{error}(z)) + 1.721(z^{-1}v_{ctrl}(z)) - 0.7558(z^{-2}v_{ctrl}(z))$$

$$v_{ctrl}[n] = 0.0182v_{error}[n-1] + 0.01657v_{error}[n-2] + 1.721v_{ctrl}[n-1] - 0.7558v_{ctrl}[n-2]$$

Apply time shifting property

$$x[n-k] = z^{-k} X(z)$$

Transfer function:

$$0.0182 z + 0.01657$$

-----  
$$z^2 - 1.721 z + 0.7558$$

Sampling time: 0.2

# 2nd-order transfer function example (digital formula from IDPOLY)

## Method #2

Rewrite discrete - time IDPOLY as

$$y(t) = \frac{0.0182q^{-1} + 0.01657q^{-2}}{1 - 1.721q^{-1} + 0.7558q^{-2}} u(t)$$

where  $y(t) \rightarrow y(k), u(t) \rightarrow u(k), q \rightarrow z$

$$y(k) = \frac{0.0182z^{-1} + 0.01657z^{-2}}{1 - 1.721z^{-1} + 0.7558z^{-2}} u(k)$$

$$y(k) = 0.0182z^{-1}u(k) + 0.01657z^{-2}u(k) + 1.721z^{-1}y(k) - 0.7558z^{-2}y(k)$$

$$y(k) = 0.0182(z^{-1}u(k)) + 0.01657(z^{-2}u(k)) + 1.721(z^{-1}y(k)) - 0.7558(z^{-2}y(k))$$

$$y[n] = 0.0182u[n-1] + 0.01657u[n-2] + 1.721y[n-1] - 0.7558y[n-2]$$

Apply time shifting property

$$x[n-k] = z^{-k} X(z)$$

As output  $y = v_{ctrl}$  and input  $u = v_{error}$

$$v_{ctrl}[n] = 0.0182v_{error}[n-1] + 0.01657v_{error}[n-2] + 1.721v_{ctrl}[n-1] - 0.7558v_{ctrl}[n-2]$$

```
Discrete-time IDPOLY model: y(t) = [B(q)/F(q)]u(t) + e(t)
```

```
B(q) = 0.0182 q^-1 + 0.01657 q^-2
```

```
F(q) = 1 - 1.721 q^-1 + 0.7558 q^-2
```

```
This model was not estimated from data.
```

```
Sampling interval: 0.2
```



# Formula Implementation in Matlab

## Matlab Implementation

```
% time vector
t=[0:0.2:12]; % sampling Tsampling is 0.2s
% initialization
error0=1; % error[n]
error1=0; % error[n-1]
error2=0; % error[n-2]
ctrl1=0; % ctrl[n-1]
ctrl2=0; % ctrl[n-2]
for i=1:length(t)
    % digital implementation of Gc(z)
    ctrl0(i)=0.0182*error1+0.01657*error2+1.721*ctrl1-
        0.7558*ctrl2;
    % store time delay data for next calculation
    error2=error1;
    error1=error0;
    error=1; % 1=step response; 0=impulse response
    ctrl2=ctrl1;
    ctrl1=ctrl0(i);
end

figure;
% plot the step response of analog and digital system
step(G); hold on; step(Gz); hold on;
% plot the response of digital implementation
plot(t,ctrl0,'ro'); hold on;
legend('G(s)', 'Gz(z)', 'Digital Implementation');
```

Initialization = 0

Calculate  $V_{ctrl}[n]$

